

"fluid kinematics"

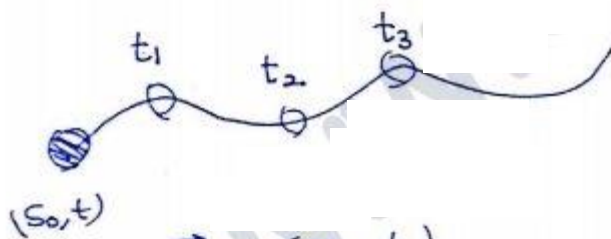
* study of motion of fluid without any ref. of force and moment.

The description of motion of fluid is defined by

① lagrangian description

② Eulerian description

lagrangian



$$\vec{s} = (s_0, t)$$

$$\vec{v} = \frac{d\vec{s}}{dt}$$

$$\vec{a} = \frac{d^2\vec{s}}{dt^2}$$

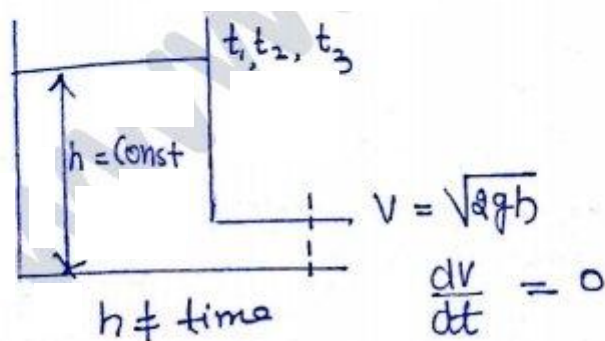
Eulerian



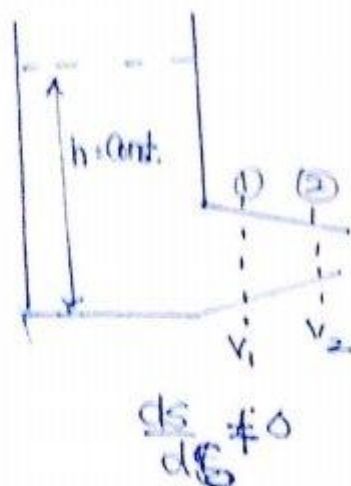
Vector $\begin{cases} \vec{v} = f(x, y, z, t) \\ \vec{a} = f(x, y, z, t) \end{cases}$

Scalar $\begin{cases} \rho = f(x, y, z, t) \\ T = f(x, y, z, t) \\ P = f(x, y, z, t) \end{cases}$

*



*



$$\vec{V} = f(\vec{x}, \vec{y}, \vec{z}, t)$$

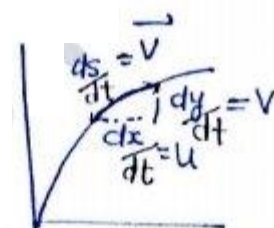
$$d\vec{V} = \frac{\partial \vec{V}}{\partial x} dx + \frac{\partial \vec{V}}{\partial y} dy + \frac{\partial \vec{V}}{\partial z} dz + \frac{\partial \vec{V}}{\partial t} dt$$

$$\vec{a} = \frac{d\vec{V}}{dt} = \frac{D\vec{V}}{Dt}$$

$$\frac{dV}{dt} = \frac{\partial \vec{V}}{\partial x} \left(\frac{dx}{dt} \right) + \frac{\partial \vec{V}}{\partial y} \left(\frac{dy}{dt} \right) + \frac{\partial \vec{V}}{\partial z} \left(\frac{dz}{dt} \right) + \frac{\partial \vec{V}}{\partial t}$$

$$\vec{V}(x, y, z, t) = u(x, y, z, t)\hat{i} + v(x, y, z, t)\hat{j} + w(x, y, z, t)\hat{k}$$

$$\frac{D\vec{V}}{Dt} = u \frac{d\vec{V}}{dx} + v \frac{d\vec{V}}{dy} + w \frac{d\vec{V}}{dz} + \frac{d\vec{V}}{dt}$$



↓
total derivative / material derivative

$$\vec{V} = u\hat{i} + v\hat{j} + w\hat{k}$$

$$\vec{\nabla} = \frac{\partial}{\partial x}\hat{i} + \frac{\partial}{\partial y}\hat{j} + \frac{\partial}{\partial z}\hat{k}$$

$$\Rightarrow \boxed{\vec{\nabla} \cdot \vec{\nabla}} = u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} + w \frac{\partial}{\partial z}$$

time space operator / ~~convective~~
Convective operator

$$\star \left[\frac{D}{Dt} = (\vec{v} \cdot \vec{\nabla}) + \frac{\partial}{\partial t} \right]$$

$$\star \left[\frac{Ds}{Dt} = (\vec{v} \cdot \vec{\nabla}) s + \frac{\partial s}{\partial t} \right]$$

$\Rightarrow \frac{\partial}{\partial t} \rightarrow$ local/temporal operators.

* for incompressible flow

$$\left[\frac{Ds}{Dt} = 0 \right]$$

$$\star \vec{a} = (\vec{v} \cdot \vec{\nabla}) \vec{v} + \frac{\partial \vec{v}}{\partial t}$$

$$(\vec{v} \cdot \vec{\nabla}) = 0$$

No acc'n $\star \left[\left(\frac{D\vec{v}}{Dt} = \vec{a} = 0 \right) \right]$

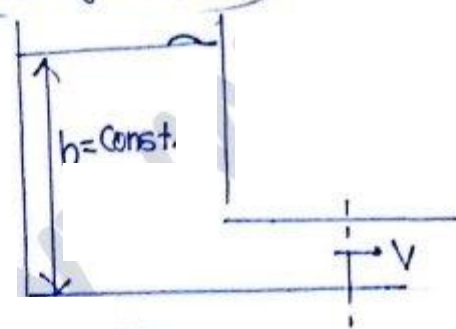
Lagrangian Description $\rightarrow$ it is an exact analysis on single fluid particle motion.

In this description single fluid particle motion is described by tracing the space vector with reference of initial point. Due to large number of variable in the system, the equation become complex.

Eulerian description: In this description study is on the bulk flow at the given section by defining the field variable in flow field. It is an approximate analysis.

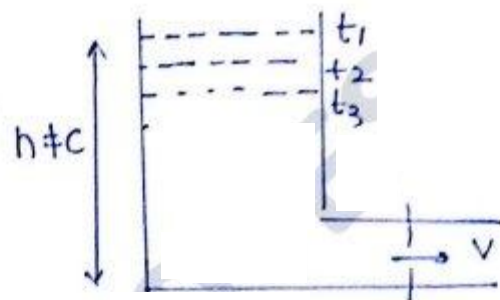
Types of fluid flow:-

* Steady & Unsteady Flow:- A flow is said to be steady when properties don't change with time, otherwise the flow is unsteady.
(at a given space)



$$h \neq f(\text{time}) \quad \frac{\partial V}{\partial t} = 0$$

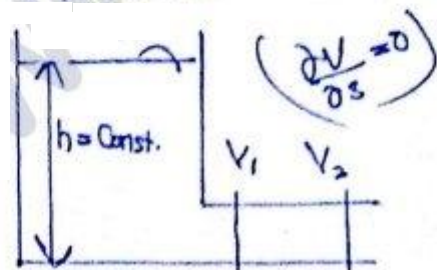
Steady flow



$$\frac{\partial V}{\partial t} \neq 0$$

Unsteady flow

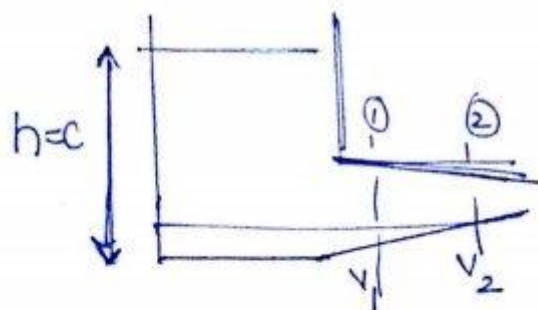
* Uniform & Non uniform Flow:- A flow is said to be uniform if the properties don't change with space at a given time, otherwise the flow is non-uniform flow.



$$A_1 V_1 = A_2 V_2 \quad (A_1 = A_2)$$

$$V_1 = V_2$$

(uniform flow)



$\Rightarrow$ Non-Uniform flow

$$A_1 \neq A_2, v_1 \neq v_2 \quad \left(\frac{\partial v}{\partial s} \right) \neq 0$$

* Laminar Flow & turbulent Flow:- When Fluid Flows in the form of laminar with negligible mixing b/w layers is known as laminar flow. $\uparrow$

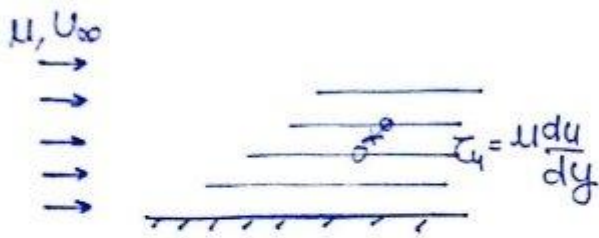
It Generally occurs at low velocity or high viscosity i.e. low Reynolds number

Laminar Flow is generally known as viscous flow. there are viscous shear stresses in flow. So laminar flow follows Newton's law of viscosity.

Turbulent Flow:- when Fluid Flow is highly disordered and there is rapid mixing b/w the layers is known as turbulent flow. It generally occurs at high velocity or low viscosity i.e. High Reynolds number. In turbulent flow due to rapid mixing b/w the layers some additional stress comes into the picture known as eddy

eddy shear stress. Their magnitude is high w.r.t. viscous stress in turbulent flow.

These stress are also known as Reynold's Stress

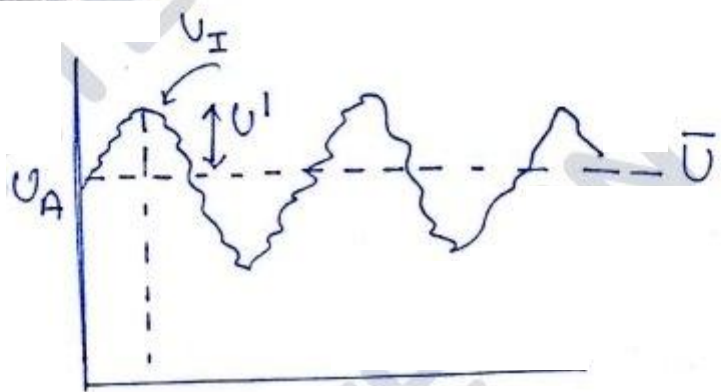
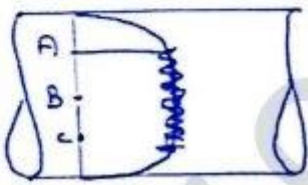


$$\downarrow Re = \frac{F_i \downarrow}{F_v \uparrow}$$

$$\downarrow Re = \frac{\rho V L}{\mu} \quad \mu \uparrow$$

In turbulent

$\Rightarrow$



Time avg. velocity

$$\bar{U} = \frac{1}{T} \int_0^T U_I dt$$

$$U = \bar{U} + U'$$

$$V = \bar{V} + V'$$

$$W = \bar{W} + W'$$

Assumption

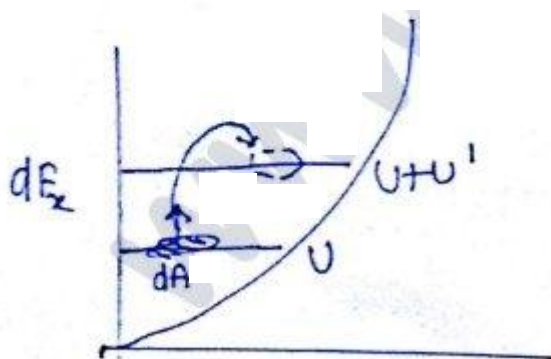
$$U' = V' = W'$$

(fluctuation)
~~fluctuation~~

$$* \quad \frac{1}{T} \int_0^T U' dt = 0$$

time avg. of fluctuation = 0

*



$$d\dot{m} = \rho dA U'$$

Net force = momentum

$$dF_x = d\dot{m}(U+U') - d\dot{m} U$$

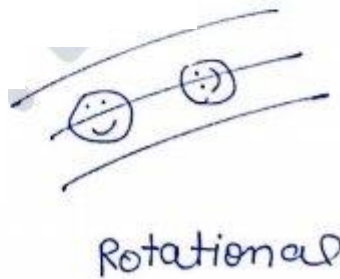
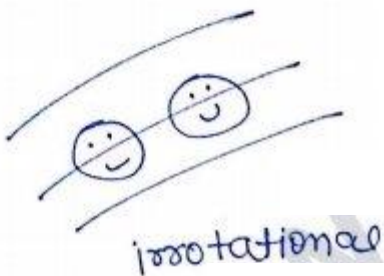
$$dF_x = d\dot{m} U'$$

$$dF_x = (\rho dA v') U'$$

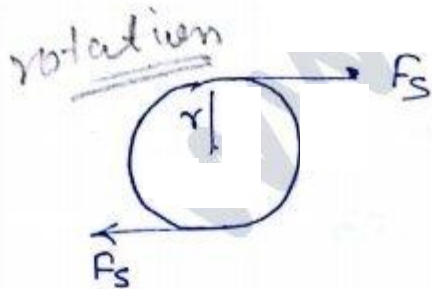
$$\frac{dF_x}{dA} = \rho v' U'$$

* $\boxed{\tau_e = \rho v' U'}$ Reynold's shear stress
or eddy stress.

Rotational and irrotational Flow :-



$u = f(x, y, z)$
 $v = f(x, y, z)$
 $w = f(x, y, z)$
 Rotational
 Flow if change
 in $\perp$ er dir'n



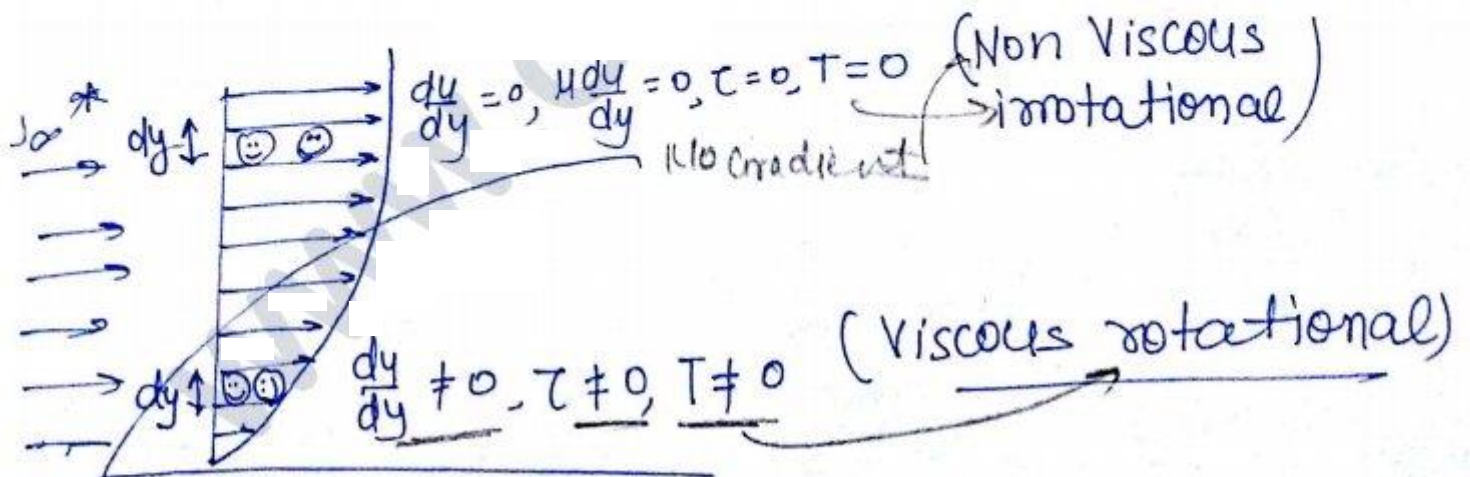
$$F_s \times r \neq 0$$

$$F_s \neq 0$$

$$\tau \times A \neq 0$$

$$\tau \neq 0 \text{ rotational}$$

$$\mu \frac{du}{dy} \neq 0 \Rightarrow \frac{du}{dy} \neq 0$$

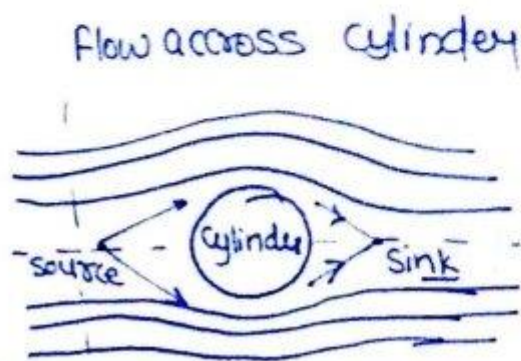
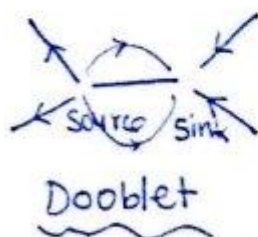
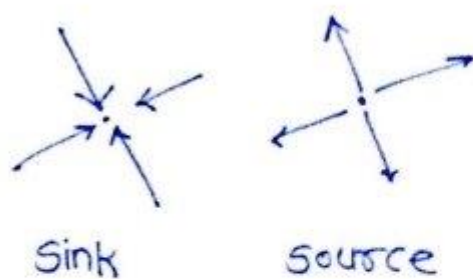
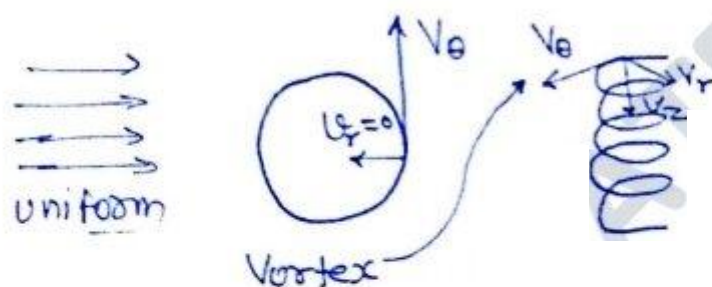


- * A Flow is said to be irrotational if there is no external torque i.e. No shear force, No Viscous shear stress, i.e. the flow is non-viscous flow.
- * One/two/three dimension flow $\rightarrow$ theory book
 {Read from}

Compressible and incompressible:- A flow is said to be incompressible

$$\frac{D\rho}{Dt} = 0 \quad \text{or} \quad \frac{d\rho}{\rho} < 5\% \quad \text{or} \quad Ma < 0.32$$

Vortex Motion:- Motion of fluid along circular path is known as vortex motion.

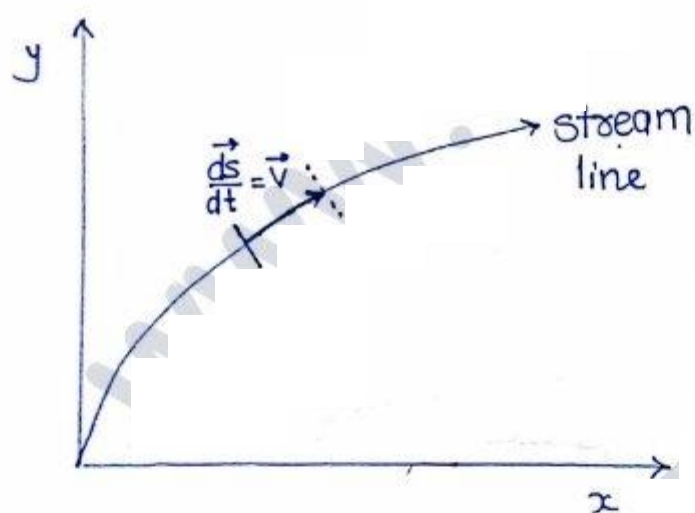


Uniform doublet

Flow lines:- There are three flow lines to define the fluid flow.

- ① stream line
- ② Path line
- ③ streak line.

Stream line:- It is an imaginary line or curve drawn in space such that the tangent drawn on the curve gives velocity vector i.e. stream line space vector and velocity vector coincides.



$$d\vec{s} = dx\hat{i} + dy\hat{j} + dz\hat{k}$$

$$\vec{V} = u\hat{i} + v\hat{j} + w\hat{k}$$

$$\boxed{d\vec{s} \times \vec{V} = 0}$$

Stream line eqⁿ.

$$\vec{A} \times \vec{B} = |\vec{A}| |\vec{B}| \sin \theta$$

$$\vec{A} \times \vec{B} = 0$$

$$\theta = 0$$

$$d\vec{s} \times \vec{V} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ dx & dy & dz \\ u & v & w \end{vmatrix} = 0$$

$$\Rightarrow (w dy - v dz)\hat{i} + (u dz - w dx)\hat{j} + (v dx - u dy)\hat{k} = 0$$

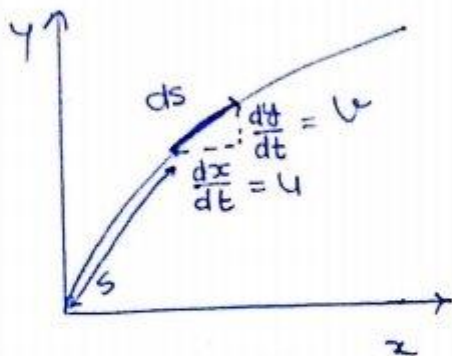
$$w dy - v dz = 0$$

$$u dz - w dx = 0$$

$$v dx - u dy = 0$$

$$* \left[\frac{dx}{u} = \frac{dy}{v} = \frac{dz}{w} \right]$$

3D eqn

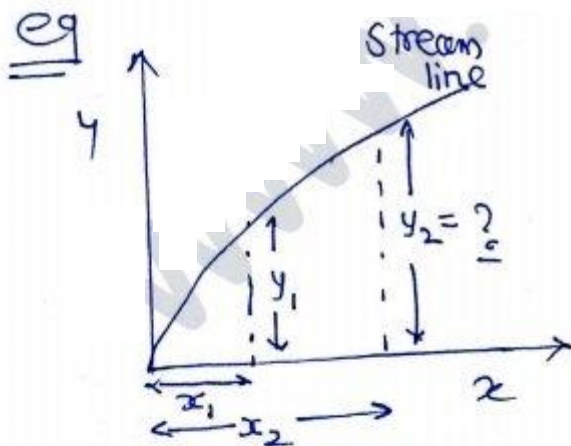


$$u = \frac{dx}{dt}, \quad v = \frac{dy}{dt}$$

$$dt = \frac{dx}{u} \quad \text{--- (1)}, \quad dt = \frac{dy}{v} \quad \text{--- (2)}$$

① & ②

$$\left[\frac{dx}{u} = \frac{dy}{v} \right]$$



$$u = 2x^2y$$

$$v = 3xy$$

$$\therefore \frac{dx}{2x^2y} = \frac{dy}{3xy}$$

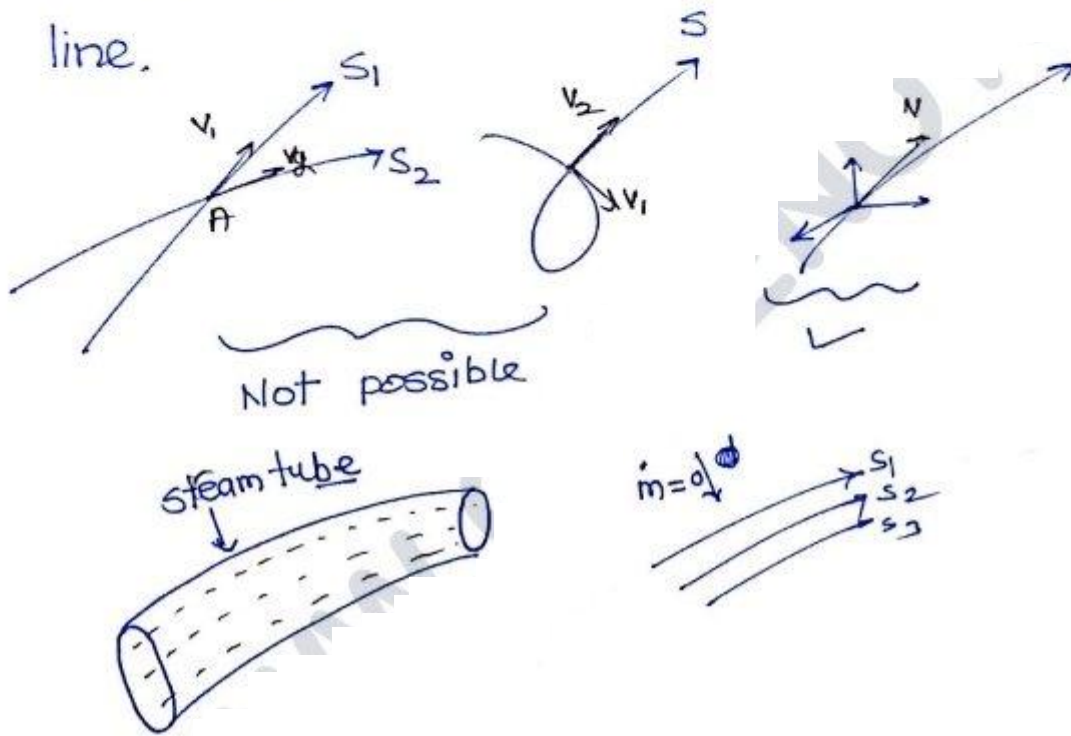
$$\int_{x_1}^{x_2} \frac{dx}{x} = \frac{2}{3} \int_{y_1}^{y_2} dy$$

$$\ln x \Big|_{x_1}^{x_2} = \frac{2}{3} (y_2 - y_1)$$

$$\ln\left(\frac{x_2}{x_1}\right) = \frac{2}{3} (y_2 - y_1)$$

$$y_2 = y_1 + \frac{3}{2} \ln\left(\frac{x_2}{x_1}\right)$$

Note:- Two stream lines will never intersect each other or one stream line will never intersect it self because at the point of intersection there will be two velocity field which is impossible so there is no mass flow rate across the stream line.

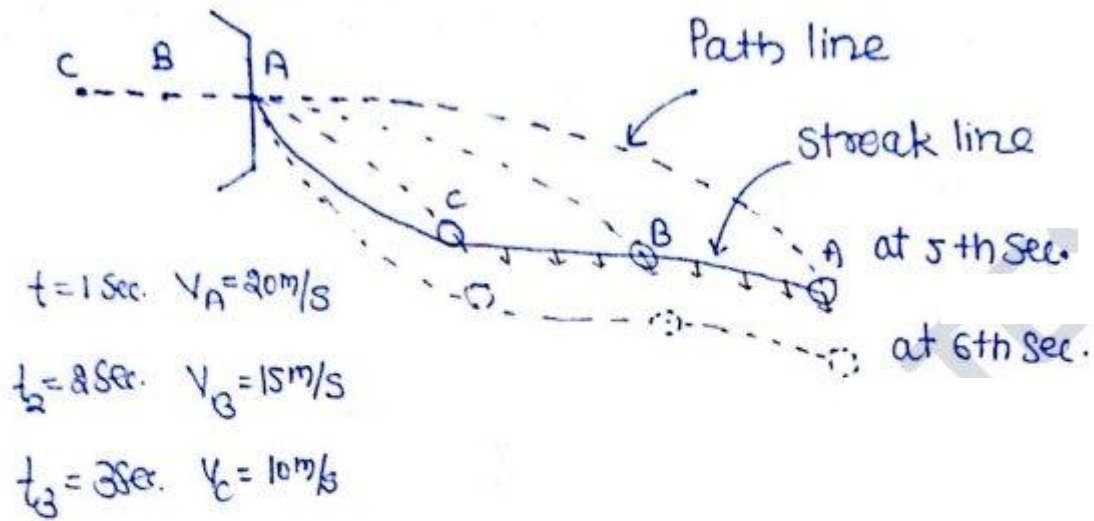


Path line:- It is the line by drawing the path of single fluid particle at diff time interval. It is defined by lagrangian description

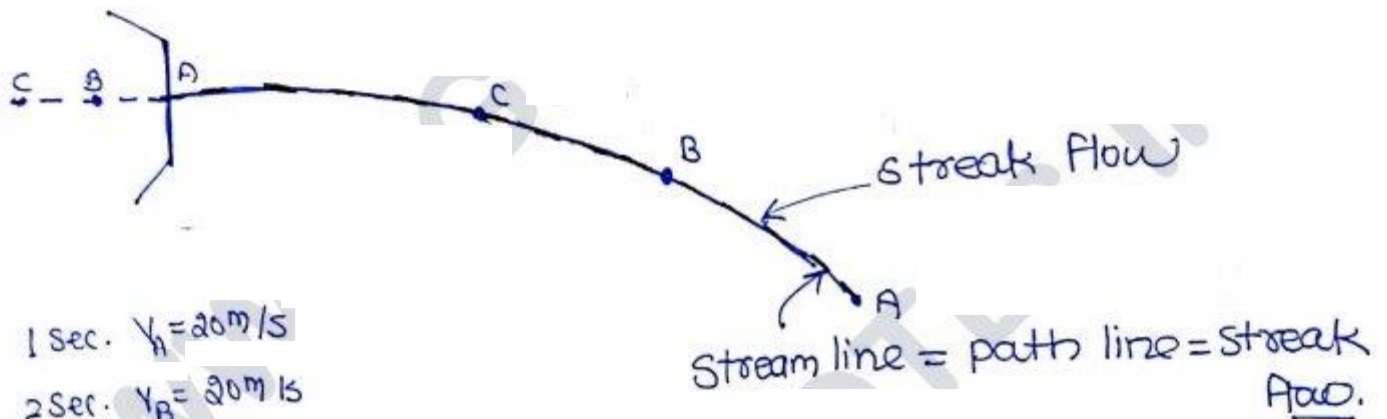


Streak line:- It is an instantaneous picture of all the particles passing through single point

* Unsteady flow



* Steady Flow.

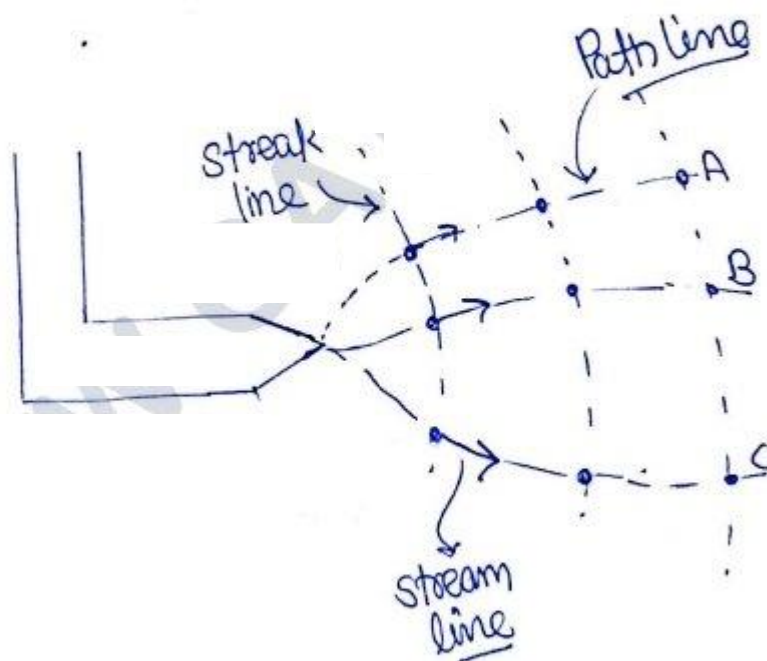


1 sec. $V_A = 20 \text{ m/s}$

2 sec. $V_B = 20 \text{ m/s}$

3 sec. $V_C = 20 \text{ m/s}$

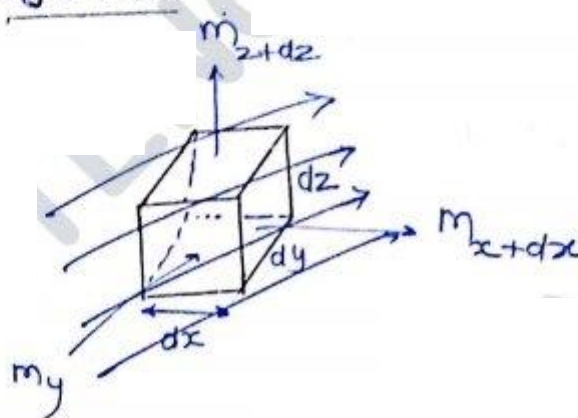
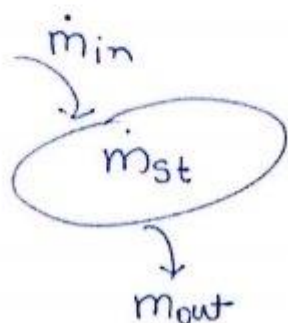
$$\frac{dv}{dt} = 0$$



- * In steady flow all three lines graphically identical
- * In unsteady flow stream line shown instantaneous picture of flow.

Continuity equation / Conservation of mass :-

In Cartesian coordinate system :-



$$\dot{m}_{in} - \dot{m}_{out} = \dot{m}_{st}$$

for x-coordinate

$$\text{LHS} \rightarrow = \dot{m}_x - \left(\dot{m}_x + \frac{\partial \dot{m}_x}{\partial x} dx \right) =$$

$$\dot{m}_x = \int (\rho dy dz u)$$

$$\rho = \rho(x, y, z, t)$$

$$u = u(x, y, z, t)$$

$$= - \frac{\partial (\rho dy dz u)}{\partial x} dx$$

$$\Rightarrow - \frac{\partial (\rho u)}{\partial x} dv \quad \text{Similarly} \quad - \frac{\partial (\rho v)}{\partial y} dv, \quad - \frac{\partial (\rho w)}{\partial z} dv$$

$$\text{RHS} \quad \dot{m}_{st} = \frac{d(\dot{m})}{dt} = \frac{d}{dt} (\rho dv) = dv \frac{d\rho}{dt}$$

$$\text{So } m_{in} - m_{out} = \dot{m}_{st}$$

$$- \frac{\partial(\rho u)}{\partial x} dv - \frac{\partial(\rho v)}{\partial y} dy - \frac{\partial(\rho w)}{\partial z} = dv \cdot \frac{\partial \rho}{\partial t}$$

$$\boxed{\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} + \frac{\partial \rho}{\partial t} = 0}$$

★ { Scalars u, v, w निकालने के लिए छोड़ें } General Continuity eqⁿ

(i) Steady Flow: $\frac{\partial \rho}{\partial t} = 0$

$$\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} = 0$$

(ii) $\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} + \frac{\partial \rho}{\partial t} = 0$

$$\rho \frac{\partial u}{\partial x} + u \frac{\partial \rho}{\partial x} + \rho \frac{\partial v}{\partial y} + v \frac{\partial \rho}{\partial y} + \rho \frac{\partial w}{\partial z} + w \frac{\partial \rho}{\partial z} + \frac{\partial \rho}{\partial t} = 0$$

$$\rho \left[\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right] + u \frac{\partial \rho}{\partial x} + v \frac{\partial \rho}{\partial y} + w \frac{\partial \rho}{\partial z} + \frac{\partial \rho}{\partial t} = 0$$

$$\rho (\vec{\nabla} \cdot \vec{v}) + (\vec{v} \cdot \vec{\nabla}) \rho + \frac{\partial \rho}{\partial t} = 0$$

★ $\boxed{\rho (\vec{\nabla} \cdot \vec{v}) + \frac{D\rho}{Dt} = 0}$

General Continuity eqⁿ

(iii) steady incompressible flow:-

$$\frac{D\rho}{Dt} = 0$$

$$\rho(\vec{\nabla} \cdot \vec{V}) = 0 \quad \rho \neq 0$$

$$\boxed{(\vec{\nabla} \cdot \vec{V}) = 0}$$

(iv) Unsteady Incompressible flow:

$$\boxed{(\vec{\nabla} \cdot \vec{V}) = 0} \quad \frac{D\rho}{Dt} = 0$$

* For incompressible flow field $(\vec{\nabla} \cdot \vec{V}) = 0$ Always

Question:- The velocity component of x & y dirⁿ are given by $U = \lambda xy^3 - x^2y$, $V = xy^2 - \frac{3}{4}y^4$ then the value of λ for incompressible fluid flow.

Sol

$$\vec{\nabla} \cdot \vec{V} = 0$$

$$\Rightarrow \frac{\partial}{\partial x}(\lambda xy^3 - x^2y) + \frac{\partial}{\partial y}(xy^2 - \frac{3}{4}y^4) = 0$$

$$\Rightarrow \lambda y^3 - 2xy + 2xy - \frac{3}{2}y^3$$

$$\lambda = 3$$

Q.5

$$u = \lambda xy^3 - x^2y$$

$$v = xy^2 - \frac{3}{4}y^4$$

$$\rho(\nabla \cdot \vec{V}) + \frac{D\rho}{Dt} = 0$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

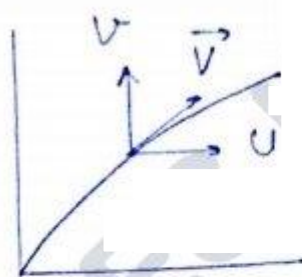
$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$(\lambda y^3 - 2xy) + (2xy - 3y^3) = 0$$

$$(\lambda - 3)y^3 = 0$$

$$\boxed{\lambda = 3} \quad 0 = 0$$

for $\lambda = 3$ we can define velocity field



Note: Each velocity field must satisfy its continuity eqⁿ otherwise the velocity field is not defined.

Q.6

$$\rho(\nabla \cdot \vec{V}) + \frac{D\rho}{Dt} = 0$$

$$1.2(\nabla \cdot \vec{V}) + 2.4 = 0$$

$$\nabla \cdot \vec{V} = -2 \text{ s}^{-1}$$

Q.7

$$\vec{V} = (5x + 6y + 7z)\hat{i} + (6x + 5y + 9z)\hat{j} + (3x + 2y + \lambda z)\hat{k}$$

$$\rho = \rho_0 \exp(-2t)$$

$$\nabla \cdot \vec{V} = (5 + 5 + \lambda)$$

$$\rho(\nabla \cdot \vec{V}) + \frac{D\rho}{Dt} = 0$$

$$\rho_0 e^{-2t} (5 + 5 + \lambda) + \rho_0 (-2) e^{-2t} = 0$$

$$\lambda + 10 - 2 = 0$$

$$\lambda = -8$$

Q.8

$$\frac{\partial}{\partial x}(\rho u) + \frac{\partial}{\partial y}(\rho v) + \frac{\partial}{\partial z}(\rho w) + \frac{\partial \rho}{\partial t} = 0$$

$$\rho \left[\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} \right] + \frac{\partial \rho}{\partial t} = 0$$

$$\rho_0 e^{-2t} (5 + 5 + \lambda) + \rho_0 (-2) e^{-2t} = 0$$

$$10 + \lambda - 2 = 0$$

$$\boxed{\lambda = -8}$$

Q.5

$$\frac{1}{x} \left(2x + \frac{\partial u}{\partial y} \right) = 0$$

$$2 + \frac{1}{x} \frac{\partial u}{\partial y} = 0$$

$$\frac{\partial u}{\partial y} = -2x$$

$$u_y = -2xy + c$$

Q.5

$$V_x = x^2$$

$$s = \frac{1}{x}$$

Steady Flow

$$\frac{\partial(su)}{\partial x} + \frac{\partial(su)}{\partial y} + \cancel{\frac{\partial s}{\partial t}} = 0$$

$$\frac{\partial}{\partial x} \left(\frac{1}{x} x^2 \right) + \frac{\partial}{\partial y} \left(\frac{u}{x} \right) + 0$$

$$1 + \frac{\partial}{\partial y} \left(\frac{u}{x} \right) = 0$$

$$V_y = -xy + f(x)$$

Q.4

incompressible $s = \text{const.}$

$$\nabla \cdot \vec{V} = 0$$

$$\frac{\partial}{\partial x} (2x^2 + z^2 + 6) + \frac{\partial}{\partial y} (y^2 + 2z^2 + 7) + \frac{\partial}{\partial z} (w) = 0$$

$$4x + 2y + \frac{\partial}{\partial z} (w) = 0$$

$$w = -4xz - 2yz + f(x, y)$$

Q.4 $\vec{V} = (x+2y+2)\hat{i} + (4-y)\hat{j}$

$$\nabla \cdot \vec{V} = 1 - 1 = 0 \text{ (incompressible)}$$

~~$\nabla \cdot \vec{V} =$~~ =

Ques Given the velocity field $V = 5x^3 \hat{i} - 15x^2y \hat{j}$
obtain the eqn for stream line

$$V = 5x^3 \hat{i} - 15x^2y \hat{j}$$

$$\frac{dx}{u} = \frac{dy}{v} = \frac{dz}{w}$$

$$\frac{dx}{5x^3} = \frac{dy}{-15x^2y}$$

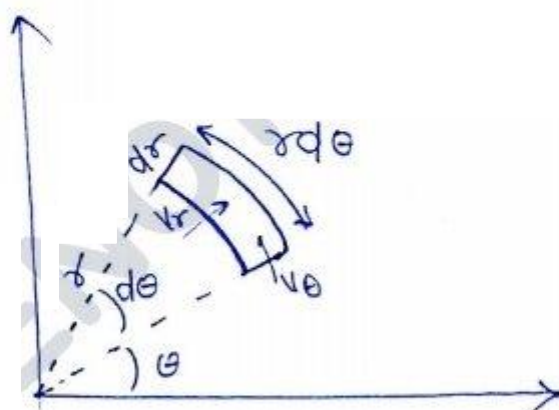
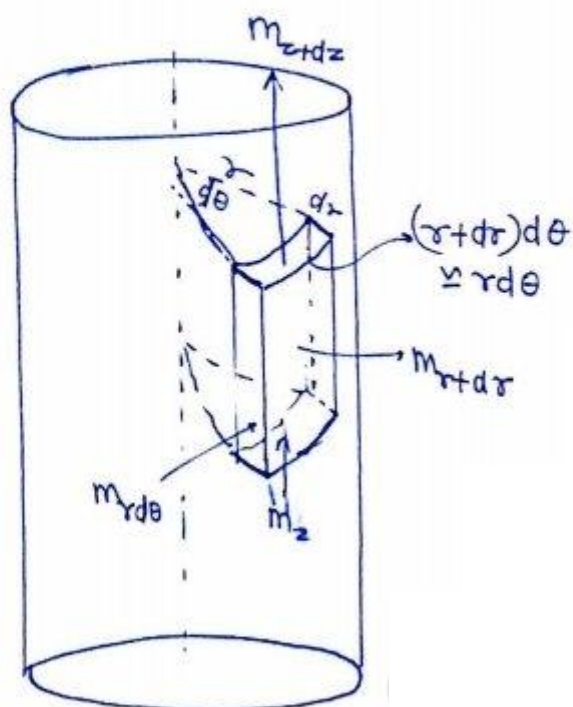
$$\frac{dy}{y} = -3 \frac{dx}{x}$$

$$\ln y = -3 \ln x + \ln c$$

$$\ln(yx^3) = \ln c$$

$$\boxed{yx^3 = c} \quad \text{or} \quad \boxed{xy^{\frac{1}{3}} = c}$$

Continuity equation in polar Coordinate System:-



$$\dot{m}_{in} - \dot{m}_{out} = \dot{m}_{sl}$$

$$\begin{array}{ccc} x & y & z \\ \downarrow & \downarrow & \downarrow \\ r & r d\theta & z \end{array}$$

$$\begin{array}{l|l} u = \frac{dx}{dt} & v_r = \frac{dr}{dt} \\ v = \frac{dy}{dt} & v_\theta = \frac{r d\theta}{dt} \\ w = \frac{dz}{dt} & v_z = \frac{dz}{dt} \end{array}$$

General Cont. eqn in polar Coordinate

$$\frac{\partial(\rho r v_r)}{\partial r} + \frac{\partial(\rho r v_\theta)}{\partial \theta} + \frac{\partial(\rho r v_z)}{\partial z} + \frac{\partial \rho}{\partial t} = 0$$

$$\frac{\partial(\rho u)}{\partial x} + \frac{\partial(\rho v)}{\partial y} + \frac{\partial(\rho w)}{\partial z} + \frac{\partial \rho}{\partial t} = 0$$

2-D incompressible flow

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$* \left[\frac{\partial(r v_r)}{r \partial r} + \frac{\partial v_\theta}{r \partial \theta} = 0 \right]$$

stream line eqⁿ

$$\frac{dx}{u} = \frac{dy}{v} \Rightarrow$$

$$* \left[\frac{dr}{v_r} = \frac{r d\theta}{v_\theta} \right]$$

if both
scalar
given
use stream
line eqⁿ

Que A leaf caught in a whirlpool At a given instant, the leaf at a distance of 120m from the centre of the whirlpool. The whirlpool can be describe by the velocity distribution

$$v_r = -\frac{60 \times 10^3}{2\pi r} \text{ m/s} \quad v_\theta = \frac{300 \times 10^3}{2\pi r} \text{ m/s.}$$

where r is the distance in meter from centre what will be the distance of leaf from centre when it has moved through half a revolution.

- (a) 48m (b) 64m (c) 120m (d) 142m

Soln

Stream line eqn $\frac{dr}{V_r} = \frac{r d\theta}{V_\theta}$

$$\frac{dr \times 2\pi r}{-60 \times 10^3} = \frac{r d\theta \times 2\pi r}{300 \times 10^3}$$

$$\frac{dr}{r} = -\frac{1}{5} d\theta$$

$$-5 \ln r = \theta + C \quad (1)$$

$$-5 \ln(120) = 0 + C$$

$$C = -5 \ln(120)$$

$$-5 \ln r = \theta - 5 \ln(120)$$

$$\theta = 5 \ln\left(\frac{120}{r}\right)$$

$$\theta = \pi$$

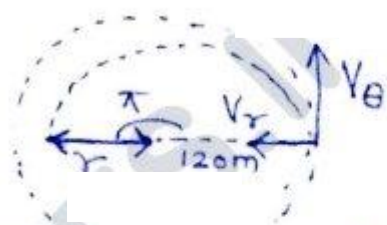
$$\pi = 5 \ln\left(\frac{120}{r}\right)$$

$$r = \frac{120}{e^{\pi/5}} = \underline{\underline{64.01 \text{ m}}}$$

$$\Rightarrow \int_{120}^r \frac{dr}{r} = -\frac{1}{5} \int_0^\pi d\theta$$

$$\ln\left(\frac{r}{120}\right) = -\frac{1}{5} \pi$$

$$\boxed{r = 64.0185 \text{ m}}$$



$$V_\theta = \frac{r d\theta}{dt}$$

$$V_r = \frac{dr}{dt}$$

$$dt = \frac{r d\theta}{V_\theta}$$

$$dt = \frac{dr}{V_r}$$

Acceleration of Fluid particle:-

In fluid flow velocity is the function of space and time so the accⁿ is the function of space and time. The space component is known as convective accⁿ and time component is local acceleration.

$$\vec{V} = f(x, y, z, t)$$

$$\vec{a} = \frac{D\vec{V}}{Dt} = \underbrace{(\vec{V} \cdot \nabla) \vec{V}}_{\text{Convective acc}^n} + \underbrace{\frac{\partial \vec{V}}{\partial t}}_{\text{local acc}^n}$$

$$\vec{a} = a_x \hat{i} + a_y \hat{j} + a_z \hat{k}$$

$$|a| = \sqrt{a_x^2 + a_y^2 + a_z^2}$$

$$a_x = \frac{du}{dt}, \quad a_y = \frac{dv}{dt}, \quad a_z = \frac{dw}{dt}$$

$$u = f(x, y, z, t)$$

$$du = \frac{\partial u}{\partial x} dx + \frac{\partial u}{\partial y} dy + \frac{\partial u}{\partial z} dz + \frac{\partial u}{\partial t} dt$$

$$a_x = \frac{du}{dt}$$

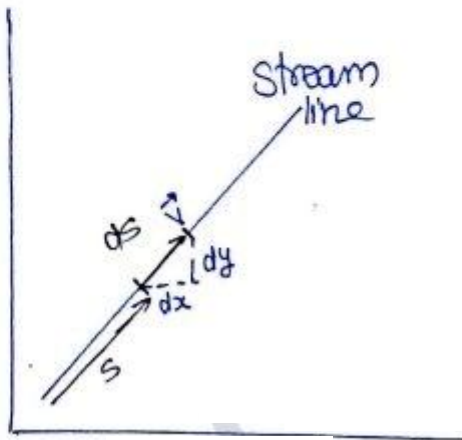
$$a_x = u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} + \frac{\partial u}{\partial t}$$

$$a_x = (\vec{V} \cdot \vec{\nabla}) u + \frac{\partial u}{\partial t}$$

$$a_y = (\vec{V} \cdot \vec{\nabla}) v + \frac{\partial v}{\partial t}$$

$$a_z = (\vec{V} \cdot \vec{\nabla}) w + \frac{\partial w}{\partial t}$$

*



$$\vec{V} = f(s, t)$$

$$\vec{a} = \left(\vec{V} \frac{\partial \vec{V}}{\partial s} + \frac{\partial \vec{V}}{\partial t} \right) \hat{s}$$

1. D

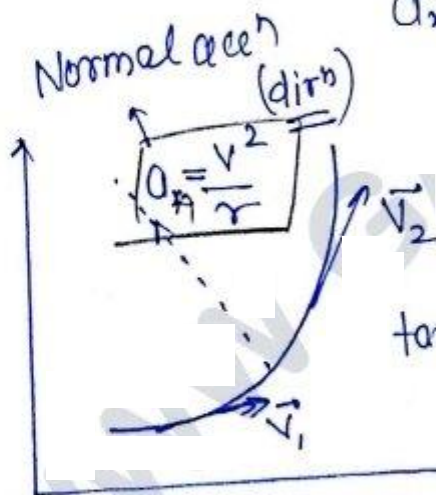
$$\frac{u}{x} \quad \frac{1}{dx}$$

$$\vec{V} = u\hat{i} + v\hat{j} + w\hat{k}$$

$$d\vec{s} = dx\hat{i} + dy\hat{j} + dz\hat{k}$$

$$\vec{a} = a_x\hat{i} + a_y\hat{j} + a_z\hat{k}$$

$$a_x = u \frac{\partial u}{\partial x} + \frac{\partial u}{\partial t}$$



tangential acc^n (mag)

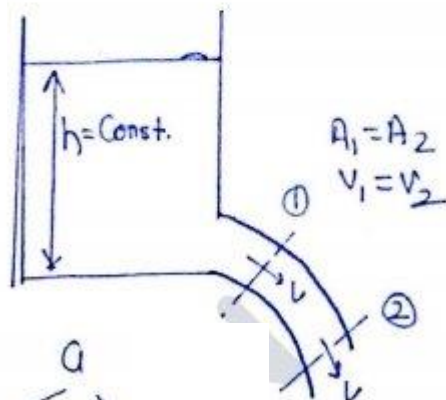
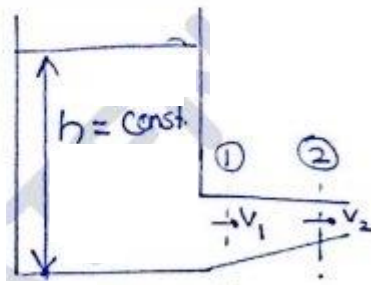
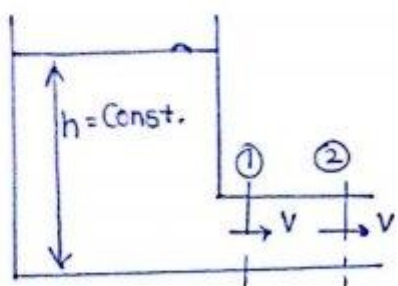
$$\vec{a}_{\text{tag}} = \left(\vec{V} \frac{dv}{ds} + \frac{dv}{dt} \right)$$

along stream line

$$\vec{a} = a_{tang} \hat{s} + a_n \hat{n}$$

$$\vec{a}_{total} = \left(v \frac{\partial v}{\partial s} + \frac{\partial v}{\partial t} \right) \hat{s} + \left(\frac{v^2}{r} \right) \hat{n}$$

* Imp



(mag.) a $a_{tang.}$ (dir'n) a_n

a a_t a_n

a a_t a_n

$$\neq a_{tc} = 0$$

$$\Rightarrow a_{nc} = 0$$

$$a_{tc} \neq 0$$

$$a_{nc} = 0$$

$$a_{tc} = 0$$

$$a_{nc} \neq 0$$

$$\Rightarrow a_{tL} = 0$$

$$\Rightarrow a_{nL} = 0$$

$$a_{tL} = 0$$

$$a_{nL} = 0$$

$$a_{tL} = 0$$

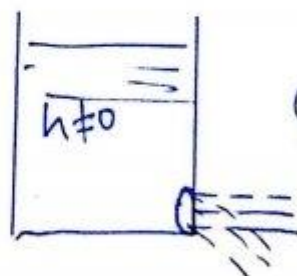
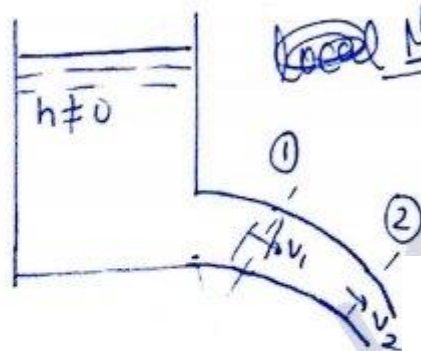
$$a_{nL} = 0$$

Convective tangential

a_{nc} / a_{tc} (b/w two points) Convective

Local Normal

a_{nL} / a_{tL} (at a point) Local



$$a_{nL} \neq 0$$

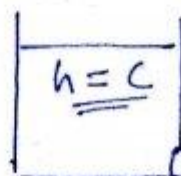
a a_t a_n

$$a_{tc} \neq 0$$

$$a_{nc} \neq 0$$

$$a_{tL} \neq 0$$

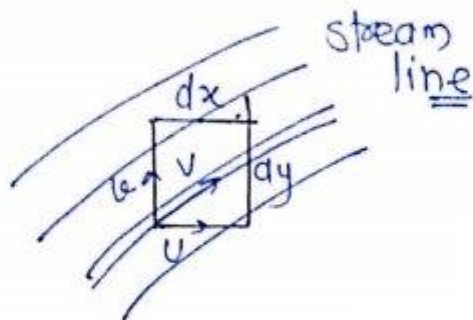
$$a_{nL} = 0$$



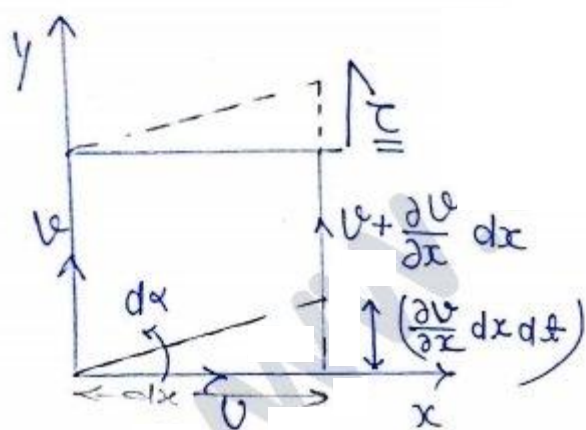
$$a_{nL} = 0$$

Rotation of Fluid Motion:-

$$\begin{aligned} u &= F(x, y) \\ v &= F(x, y) \end{aligned} \quad \left\{ \begin{array}{l} \text{(variation in } \perp \text{ dirn)} \\ \text{Rotation} \end{array} \right.$$



$$\omega_{\text{mean}} = \frac{\omega_1 + (-\omega_2)}{2} = \frac{\omega_1 - \omega_2}{2}$$

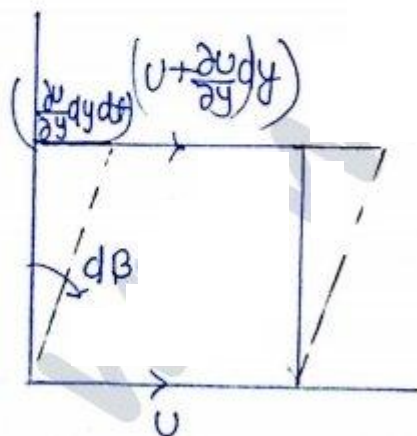


$$\tan(d\alpha) = \frac{\frac{\partial u}{\partial x} dx dt}{dx}$$

Small angle

$$d\alpha = \frac{\frac{\partial u}{\partial x} dx dt}{dx}$$

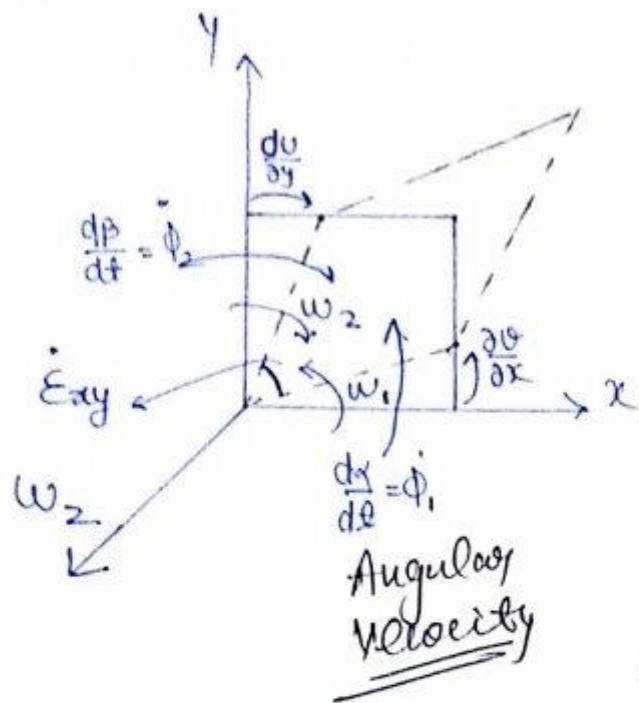
$$\left(\omega = \frac{d\theta}{dt} \right) \quad \text{Rate of Shear strain} \quad \omega_1 = \frac{d\alpha}{dt} = \frac{\partial u}{\partial x} \quad \uparrow \text{ (ccw) } \underline{+ve}$$



Similarly

$$\frac{d\beta}{dt} = \frac{\partial u}{\partial y}$$

$$\omega_2 = \frac{\partial u}{\partial y} \quad \downarrow \underline{-ve}$$



$$\omega_{\text{mean}} = \frac{\omega_1 + (-\omega_2)}{2} = \omega_z$$

$$\omega_z = \frac{\omega_1 - \omega_2}{2}$$

$$\omega_z = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$

Angular Velocity

$$\dot{\epsilon}_{xy} = \frac{|\phi_1| + |\phi_2|}{2}$$

Rate of
shear
strain

$$\dot{\epsilon}_{xy} = \frac{1}{2} \left[\left| \frac{dv}{dx} \right| + \left| \frac{du}{dy} \right| \right]$$

Rate of shear strain/deformation

Q.22 $u = 6y, \quad v = 0$

$$\omega_z = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) = \frac{1}{2} (0 - 6) = -3$$

$$\dot{\epsilon}_{xy} = \frac{1}{2} (0 + 6) = 3$$

Note:-

Rotationality

$$\omega = \frac{1}{2} (\text{curl } \vec{V})$$

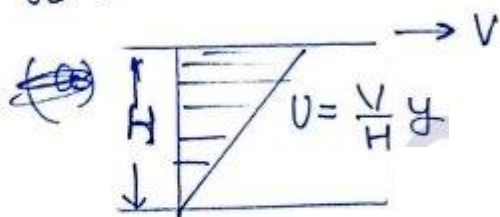
$$\omega = \frac{1}{2} (\vec{\nabla} \times \vec{V})$$

$$\omega = \frac{1}{2} \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ u & v & w \end{vmatrix}$$

$$\omega = \frac{1}{2} \left(\frac{\partial w}{\partial y} - \frac{\partial v}{\partial z} \right) \hat{i} + \frac{1}{2} \left(\frac{\partial u}{\partial z} - \frac{\partial w}{\partial x} \right) \hat{j} + \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) \hat{k}$$

$$\omega = \omega_x \hat{i} + \omega_y \hat{j} + \omega_z \hat{k}$$

Ques The laminar flow takes place b/w closely spaced parallel plate as shown in fig. Velocity profile is given by $u = \frac{V}{H} y$. The rate of rotation of fluid particles is given.



$$\vec{V} = u \hat{i} + 0 \hat{j} + 0 \hat{k}$$

$$\vec{V} = \frac{V}{H} y \hat{i}$$

$$\omega = \frac{1}{2} \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ \frac{V}{H} y & 0 & 0 \end{vmatrix}$$

$$\omega = \frac{1}{2} (0-0) \hat{i} - \frac{1}{2} (0-0) \hat{j} + \frac{1}{2} \left(0 - \frac{V}{H} \right) \hat{k} = 0 \hat{i} + 0 \hat{j} - \frac{V}{2H} \hat{k}$$

irrotational flow :-

$$\omega = 0, \omega_x = \omega_y = \omega_z = 0$$

$$\frac{1}{2}(\vec{\nabla} \times \vec{V}) = 0$$

$$\text{curl } \vec{V} = 0$$

Q.50

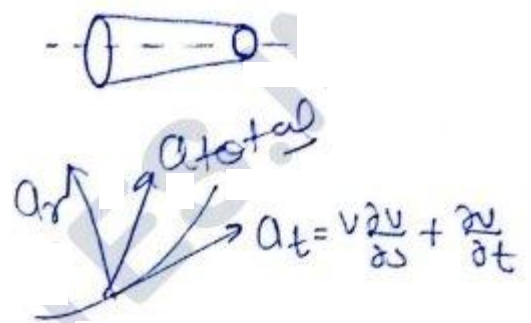
$$V = 2t \left(1 - \frac{x}{2L} \right)^2$$

$$V = f(x, t)$$

$$a_n = \text{Normal acc'n} \\ = 0$$

$$a_{\text{total}} = V \left(\frac{\partial V}{\partial s} \right) + \left(\frac{\partial V}{\partial t} \right)$$

$$= \underbrace{V \left(\frac{\partial V}{\partial x} \right)}_{\text{Conv.}} + \underbrace{\frac{\partial V}{\partial t}}_{\text{local}}$$



$$(i) a_c = V \frac{\partial V}{\partial x} = 2t \left(1 - \frac{x}{2L} \right)^2 \left[2t \times 2 \left(1 - \frac{x}{2L} \right) \left(-\frac{1}{2L} \right) \right]$$

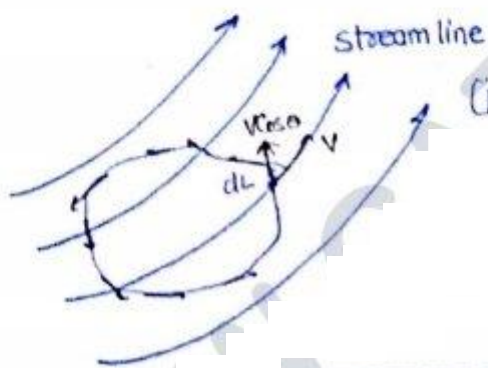
$$a_c = -14.623 \text{ m/s}^2$$

$$(ii) a_{\text{local}} = \frac{\partial V}{\partial t} = 2 \left[1 - \frac{x}{2L} \right] =$$

$$(iii) a_{\text{ho}} = -13.62 \text{ m/s}^2$$

Circulation:-

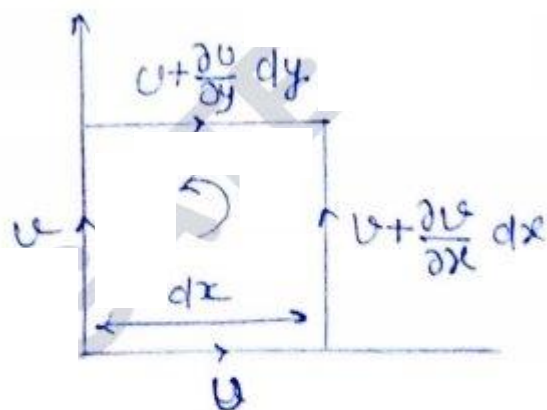
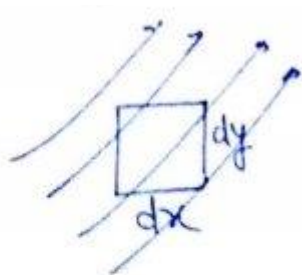
- It is the line integration of tangential component of velocities along closed loop.
- It is measure of rotationality in the flow.
- It does not depend on shape of the loop.
- Counter cw movement should be considered +ve.



$$\text{Circulation } \Gamma = \oint V \cos \theta \cdot dL$$

$$\Gamma = \oint \vec{V} \cdot d\vec{L}$$

$$\oint_L \vec{V} \cdot d\vec{L} = \iiint_A (\vec{\nabla} \times \vec{V}) \cdot \hat{n} dA$$



$$\Gamma_{ABCD} = \Gamma_{AB} + \Gamma_{BC} + \Gamma_{CD} + \Gamma_{DA}$$

$$= U dx \cos 0^\circ + \left(U + \frac{\partial U}{\partial x} dx \right) dy \cos 0^\circ + \left(U + \frac{\partial U}{\partial y} dy \right) dx \cos 180^\circ + U dy \cos 180^\circ$$

$$\Gamma_{ABCD} = \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) dx dy$$

Circulation $\Gamma_{ABCD} = \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) dA$

$$\frac{\Gamma_{ABCD}}{dA} = \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$

$$\frac{\Gamma_{ABCD}}{dA} = \Omega \quad (\text{Vorticity})$$

$$\Omega = 2\omega$$

$$\Omega = \text{curl } \vec{V} = \vec{\nabla} \times \vec{V}$$

$$\Omega = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ u & v & w \end{vmatrix} = \Omega_x \hat{i} + \Omega_y \hat{j} + \Omega_z \hat{k}$$

Vorticity:- Vorticity is the mathematical measure of rotationality. It is defined as Circulation per Area. It is twice of Rotation.

The dirⁿ of vorticity is same as dirⁿ of rotation

$$\Omega = 2\omega$$

Q.24

$$\Gamma = \iint (\nabla \times \vec{V}) \cdot \hat{n} \, dA$$

~~$\nabla \times \vec{V}$~~

$$\Gamma = \Omega \times A$$

$$= \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) A$$

$$\Gamma = [0-3] \pi 2^2$$

$$\Gamma = -12\pi \text{ Unit}$$

Q.25

$$\vec{V} = k(y\hat{i} + x\hat{k})$$

$$\Omega = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ ky & 0 & kx \end{vmatrix}$$

$$\Omega = (0-0)\hat{i} - j(k-0) + (0-k)\hat{k}$$

$$\Omega = -k\hat{j} - k\hat{k}$$

$$\boxed{\Omega_z = -k} \text{ Ans}$$

Velocity potential function

$$\phi = f(x, y, z, t)$$

- * Scalar function
- * Function of space & time
- * it is defined for irrotational flow
- * it is defined in such a way the -ve derivative of potential function in any direction gives velocity component in that dirn

$$+ dx > 0, d\phi < 0$$

$$\left(-\frac{d\phi}{dx}\right) > 0$$

$$-\frac{\partial \phi}{\partial x} = u, -\frac{\partial \phi}{\partial y} = v, -\frac{\partial \phi}{\partial z} = w$$

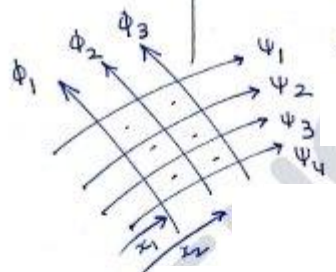
steady, incompressible flow

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

Stream function

$$\psi = f(x, y, z, t)$$

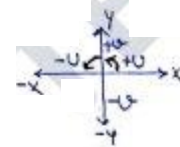
- * It is the scalar function
- * Function of space & time
- * it is defined for Continuity equation i.e. for Conservation of mass.
- * The derivative of stream function in any direction gives velocity scalar in per dirn (ccw and cw)



$$dx > 0$$

$$\phi_1 > \phi_2 > \phi_3$$

$$d\phi < 0$$



$$\frac{\partial \psi}{\partial x} = v$$

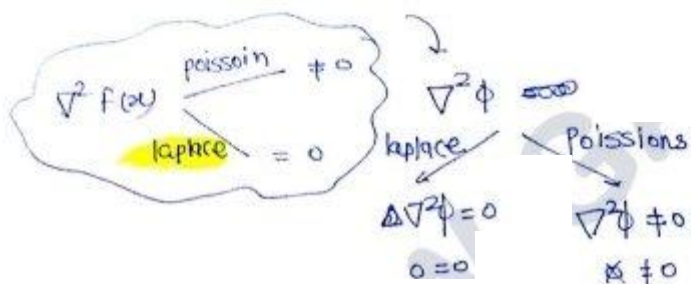
$$\frac{\partial \psi}{\partial y} = -u$$

2D, steady incompressible.

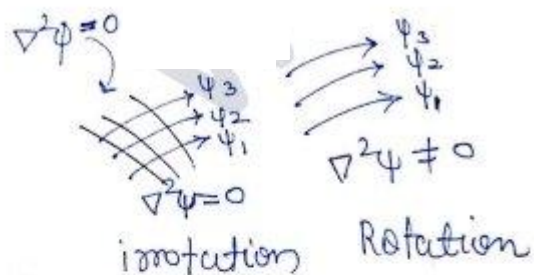
$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$-\frac{\partial^2 \phi}{\partial x^2} - \frac{\partial^2 \phi}{\partial y^2} - \frac{\partial^2 \phi}{\partial z^2} = 0$$

$$\frac{\partial^2 \phi}{\partial x^2} + \frac{\partial^2 \phi}{\partial y^2} + \frac{\partial^2 \phi}{\partial z^2} = 0, \nabla^2 \phi = 0$$



$$\nabla^2 \phi = 0 \text{ (only defined for irrotational)}$$



$$\frac{\partial}{\partial x} \left(-\frac{\partial \psi}{\partial y} \right) + \frac{\partial}{\partial y} \left(\frac{\partial \psi}{\partial x} \right) = 0$$

$$-\frac{\partial^2 \psi}{\partial x \partial y} + \frac{\partial^2 \psi}{\partial x \partial y} = 0$$

$$0 = 0$$

$$\text{Rotationality } \omega_z = \frac{1}{2} \left[\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right]$$

$$\omega_z = \frac{1}{2} \left[\frac{\partial}{\partial x} \left(\frac{\partial \psi}{\partial x} \right) - \frac{\partial}{\partial y} \left(-\frac{\partial \psi}{\partial y} \right) \right]$$

$$\omega_z = \frac{1}{2} \left[\frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} \right]$$

$$\omega_z = \frac{1}{2} (\nabla^2 \psi)$$

$$\text{Laplace } \nabla^2 \psi = 0 \rightarrow \omega_z = 0$$

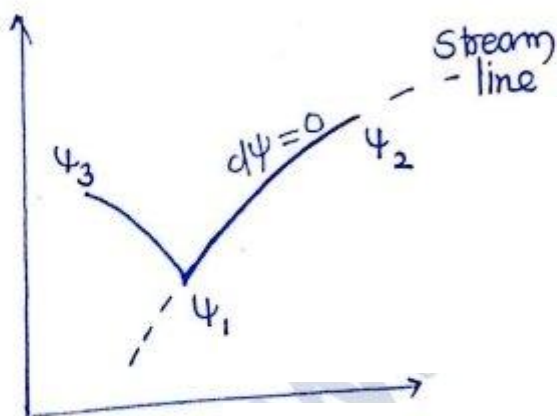
irrotational

$$\text{Poisson } \nabla^2 \psi \neq 0 \rightarrow \omega_z \neq 0$$

rotational

- * Potential function (ϕ) defined for irrotational flow so irrotational flow is also known as potential flow.
- * Potential flow is defined for 3-D flow.
- * If potential function satisfied laplace eqⁿ then potential function is valid
- * Stream function (ψ) defined for both rotational & irrotational flow
- * It is defined for 2-D flow
- * If ψ satisfied laplace eqⁿ then flow is irrotational.

Physical Significance of stream function (ψ):-



$$\psi = F(x, y)$$

$$d\psi = \frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy$$

$$d\psi = v dx - u dy \quad \because \frac{dx}{u} = \frac{dy}{v}$$

$$\boxed{d\psi = 0} \text{ along stream line.}$$

Stream function is the point function. The locus of constant stream functions is the stream line i.e. for the given stream line there is a given stream function.

* The discharge per unit depth in flow field is defined by

$$Q_{12} = |\psi_1 - \psi_2|$$

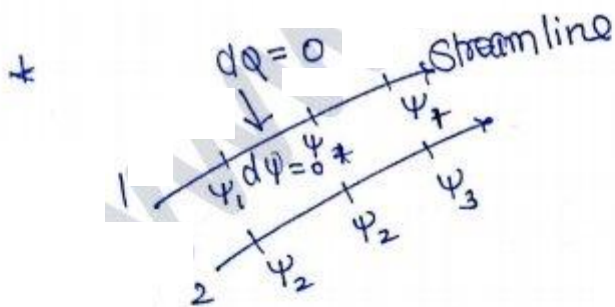
$$\Rightarrow d\psi = v dx - u dy$$

$$\frac{dx}{u} = \frac{dy}{v} \quad \text{stream line}$$

$$v dx - u dy = 0$$

$$d\psi = 0 \quad \text{along stream line}$$

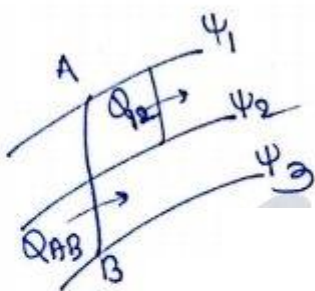
$$\boxed{\int d\psi = 0} \quad \boxed{\psi = \text{Const.}}$$



$$|d\phi| = |d\psi|$$

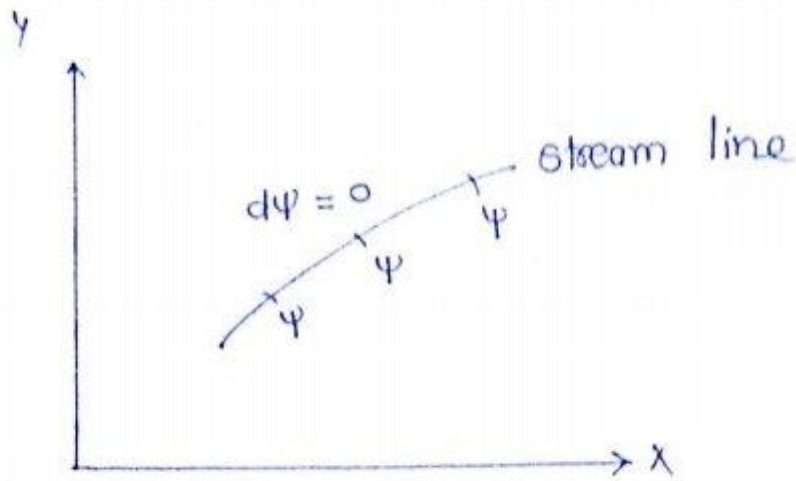
↓
discharge per unit depth

$$Q_{12} = |\psi_1 - \psi_2|$$



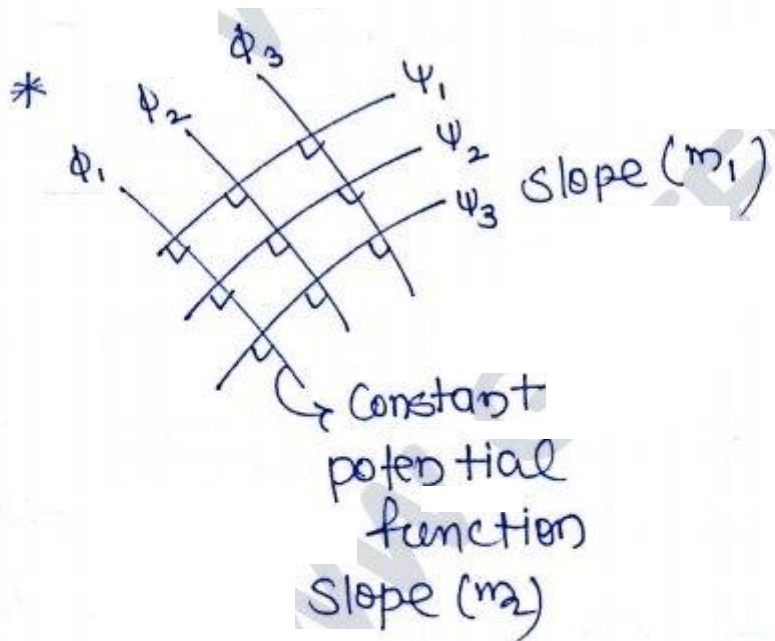
$$\boxed{Q_{AB} = |\psi_A - \psi_B|}$$

Note:-



Slope of Stream line $\Rightarrow \frac{dx}{u} = \frac{dy}{v}$, $m_1 = \frac{v}{u}$

$$\frac{dy}{dx} = \frac{\left(\frac{d\psi}{dx}\right)}{\left(-\frac{\partial\psi}{\partial y}\right)}$$



$$m_1 \times m_2 = -1$$

$$m_2 = -\frac{1}{m_1} = -\frac{u}{v}$$

$$m_2 = -\frac{u}{v}$$

Cauchy - Riemann eqn:- (C-R eqn)

$$-\frac{\partial \phi}{\partial x} = u$$

$$-\frac{\partial \phi}{\partial y} = v$$

$$\frac{\partial \psi}{\partial x} = u$$

$$\frac{\partial \psi}{\partial y} = -v$$

*

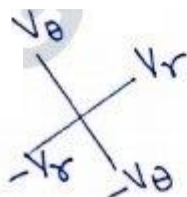
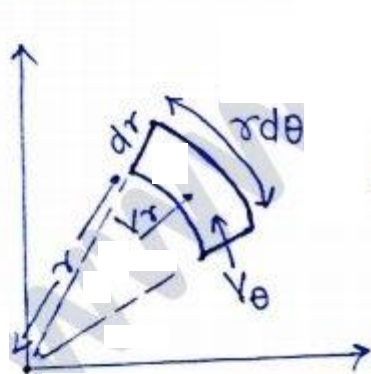
$$\frac{\partial \phi}{\partial x} = \frac{\partial \psi}{\partial y}$$

*

$$\frac{\partial \phi}{\partial y} = -\frac{\partial \psi}{\partial x}$$

Valid for
irrotational
flow

In polar Coordinates:-



$$-\frac{\partial \phi}{\partial r} = v_r$$

$$-\frac{\partial \phi}{\partial \theta} = v_\theta$$

$$\frac{\partial \psi}{\partial r} = v_\theta$$

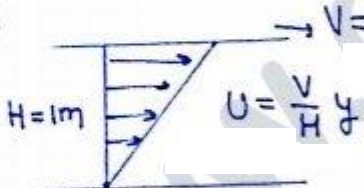
$$\frac{\partial \psi}{\partial \theta} = -v_r$$

$$\frac{\partial \phi}{\partial r} = \frac{\partial \psi}{\partial \theta}$$

$$-\frac{\partial \phi}{\partial \theta} = \frac{\partial \psi}{\partial r}$$

Q 34

p.g 28
WB



$$v = 75 \text{ m/s}$$

$$U = \frac{75}{1} y$$

$$U = 75 y$$

$$V = 0 \text{ (assumed)}$$

$$\frac{\partial \psi}{\partial x} = u = 0$$

$$0 + f'(x) = 0$$

$$f(x) = \underline{\underline{C}}$$

$$\frac{\partial \psi}{\partial y} = -U = -75 y$$

$$\partial \psi = -75 y dy$$

$$\psi = -37.5 y^2 + \underline{\underline{f(x)}}$$

$$\psi = -37.5 y^2 + C$$

Now find $\phi = ?$

$$-\frac{\partial \phi}{\partial x} = u$$

$$\frac{\partial \phi}{\partial x} = -15y$$

$$\phi = -15xy + f(y)$$

$$-\frac{\partial \phi}{\partial y} = v = 0$$

$$\frac{\partial \phi}{\partial y} = -15x + f'(y) = 0$$

$$f'(y) = 15x$$

or by C-R eqn

$$\frac{\partial \phi}{\partial x} = \frac{\partial \phi}{\partial y}$$

$$\frac{\partial \phi}{\partial x} = -15y$$

$$\phi = -15xy + C$$

wrong

because $u = 15y$

ϕ can't be
determined

Flow is rotational

Ques

$$\psi = m \ln x$$

$$-\frac{\partial \phi}{\partial x} = \frac{\partial \psi}{\partial x}$$

$$-\frac{\partial \phi}{\partial x} = \frac{m}{x} \Rightarrow \phi = m \theta + C$$

Vortex Motion:-

Forced vortex:-

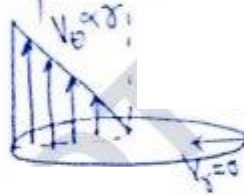
$$\tau \neq 0$$

rotational

$$\begin{cases} \psi \checkmark \\ \phi \times \end{cases}$$

$$V_\theta = r\omega$$

$$V_r = 0$$



$$\omega_z = \frac{1}{2} \left(\frac{\partial(rV_\theta)}{\partial r} - \frac{\partial V_r}{\partial \theta} \right)$$

$$\omega_z = \frac{1}{2} \left(\frac{\partial(r^2\omega)}{\partial r} - 0 \right)$$

$$\omega_z = \frac{1}{2} \cdot \omega \cdot 2 = \omega$$

$$\boxed{\omega_z = \omega} \quad \text{Rotational}$$

Thick

$$\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} = 0$$

$$\omega_z = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right) = 0$$

$$\frac{\partial(rV_r)}{\partial r} + \frac{\partial(rV_\theta)}{\partial \theta} = 0$$

$$\omega_z = \frac{1}{2} \left(\frac{\partial(rV_\theta)}{\partial r} - \frac{\partial V_r}{\partial \theta} \right)$$

Free Vortex $T = 0$

$$f \times r = 0$$

$$\begin{cases} \psi \checkmark \\ \phi \checkmark \end{cases}$$

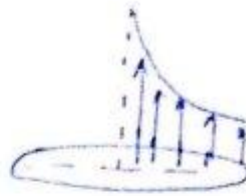
$$\frac{d(mv)}{dt} \times r = 0$$

$$\frac{d}{dt}(mvr) = 0$$

$$mV_\theta r = C$$

$$V_\theta r = C$$

$$V_\theta = \frac{C}{r}$$



$$r \rightarrow 0, V_\theta \rightarrow \infty$$

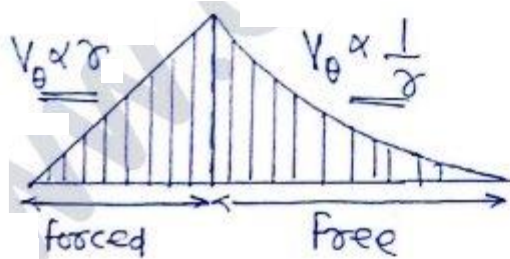
$$\omega_z = \frac{1}{2} \left(\frac{\partial(rV_\theta)}{\partial z} - \frac{\partial V_\theta}{\partial \theta} \right)$$

$$\omega_z = \frac{1}{2} \left(\frac{\partial C}{\partial z} \right)$$

$$\omega_z = 0$$

irrotation flow

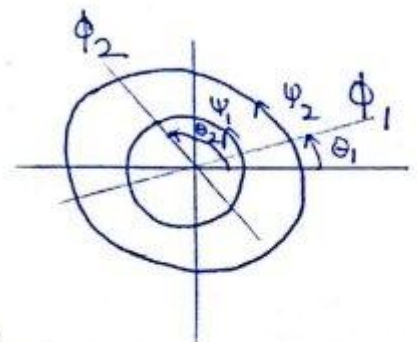
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Free vortex $\begin{cases} \phi \\ \psi \end{cases}$

$$V_\theta = \frac{C}{r}$$

$$V_r = 0$$



$$-\frac{\partial \phi}{\partial \theta} = V_\theta$$

$$\frac{\partial \psi}{\partial r} = V_\theta$$

$$-\frac{\partial \phi}{\partial r} = \frac{C}{r}$$

$$\frac{\partial \psi}{\partial r} = \frac{C}{r}$$

$$\boxed{\phi = -C\theta + C_1}$$

with θ

$$\boxed{\psi = C \ln r + C_2}$$

with radius

Sink/Source

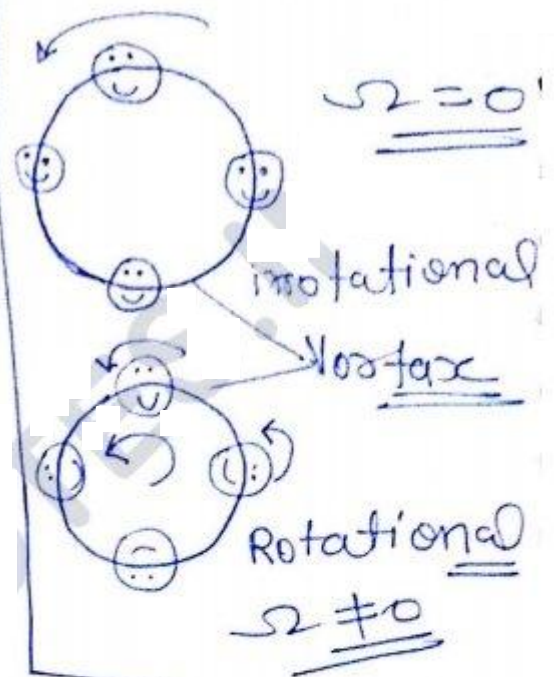
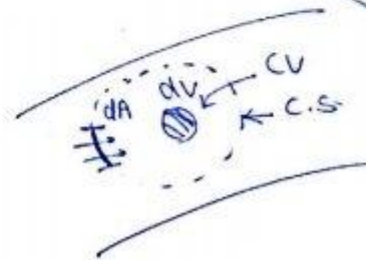
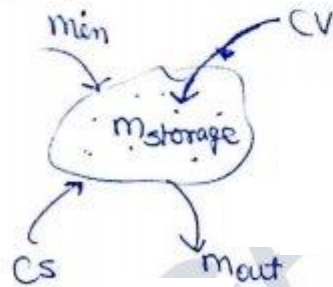
$$V_r \neq 0$$

$$V_\theta = 0$$

$$\phi = c \ln r + c_2$$

$$\psi = -c\theta + c_1$$

Continuity eqⁿ



$$\frac{dM}{dt} = \int_{C.S.} \rho V dA + \frac{d}{dt} \int_{CV} \rho dv \quad m = \text{kg}$$

$\text{kg/s} \quad \text{1/s}$

*

$$\int_{C.S.} \rho V dA + \frac{d}{dt} \int_{CV} \rho dv = 0$$
 Continuity eqⁿ

Now

$$\dot{\mathbf{p}} = \dot{m} \cdot \mathbf{v} \quad \text{momentum}$$

*

$$\dot{\mathbf{p}} = \int_{C.S.} \rho V^2 dA + \frac{d}{dt} \int_{CV} \rho \mathbf{V} \cdot d\mathbf{v}$$