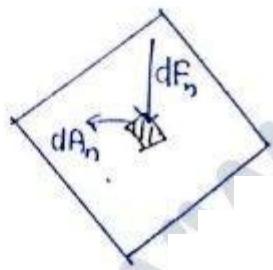


Fluid static

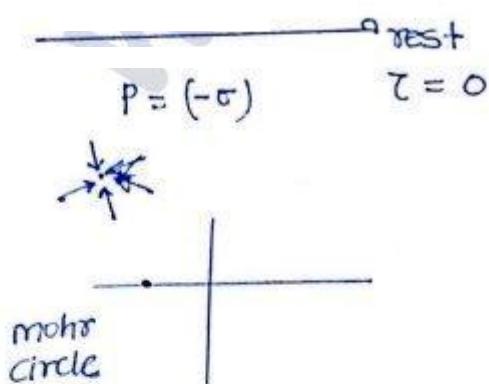
Pressure Measurement:-

Pressure :- Normal force per unit area exerted by the fluid known as pressure.

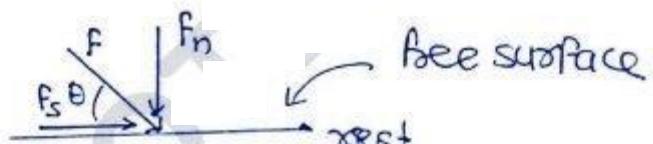
pressure is compressive in nature as per the pascal when the fluid is at rest the pressure has no direction i.e. infinite direction. So pressure is scalar in nature.



$$P = \frac{dF_n}{dA} \quad \text{N/m}^2 = \text{Pa}$$



* Always act \perp



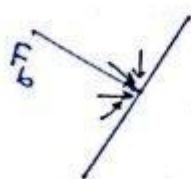
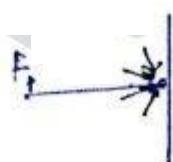
$$F_n = P \sin \theta \quad \left. \begin{array}{l} \text{it is not necessary} \\ \text{that there will be no} \\ \text{shear stress but} \\ \text{normal stress must be 0} \end{array} \right\}$$

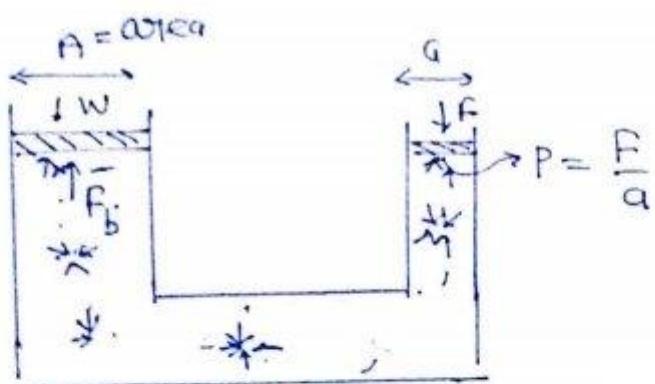
$$F_s = F \cos \theta$$

$\text{Free Surface} \quad \tau = 0$

$$F_s = 0, \cos \theta = 0$$

$$\theta = 90^\circ$$





$$F_p = P \times A$$

$$P \times A = W$$

$$F_p = W$$

$$\frac{F}{A} \times A = W \Rightarrow \frac{W}{F} = \frac{A}{a}$$

$A > a$

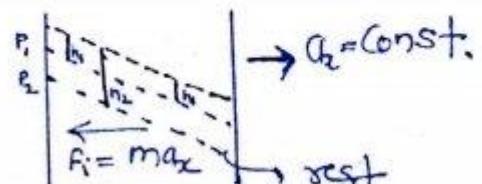
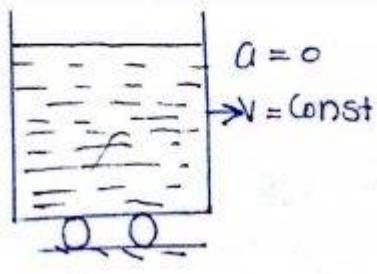
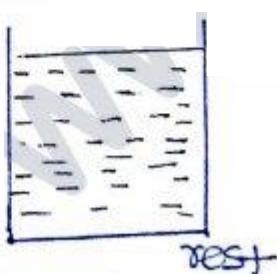
$$W > F$$

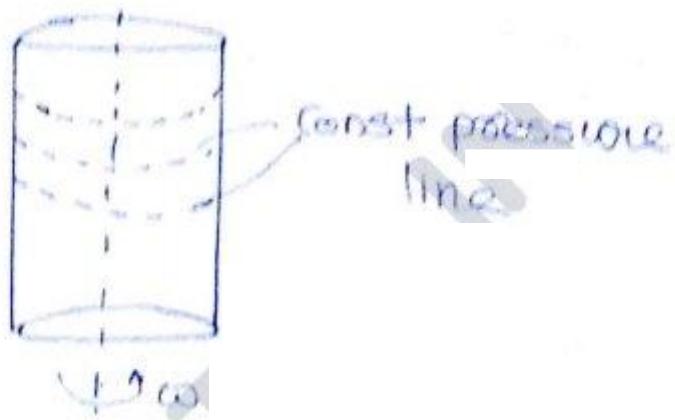
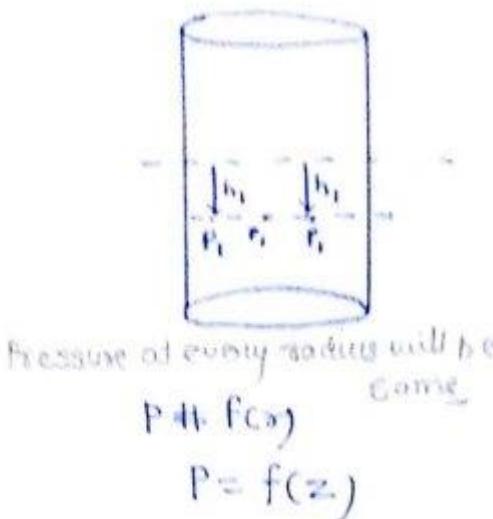
Validity of Pascal's law:-

(i) It is valid for compressible, incompressible, viscous, non viscous fluid but the fluid must be at rest.

(ii) If the fluid in a container, container is moving with constant velocity, uniform accn and constant angular velocity.

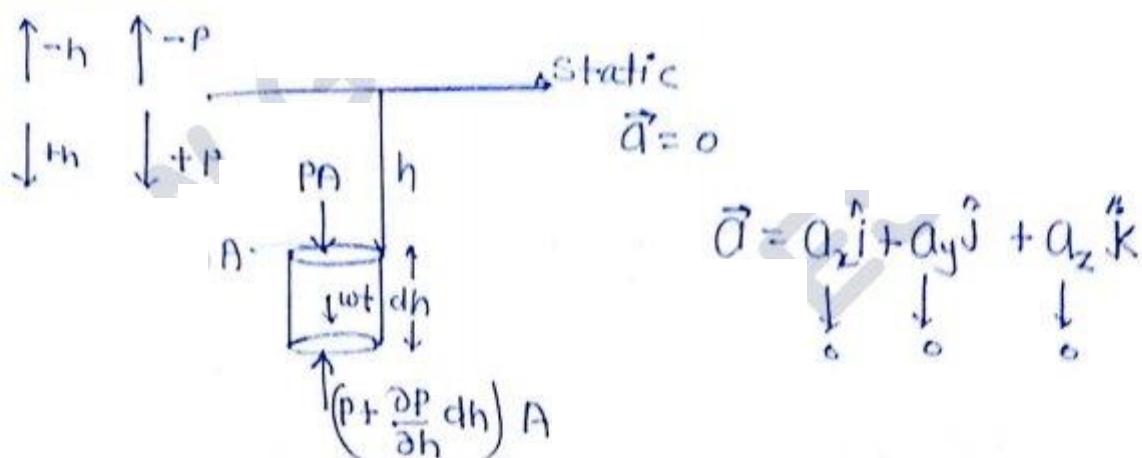
(iii) Fluid is ideal.





Hydrostatic Load

our convention



$$\sum F_{\text{net}} = m \ddot{a}_x$$

$$PA + \rho g A dh - \left(P + \frac{\partial P}{\partial h} dh \right) A = 0$$

$$PA + \rho g A dh - PA - \left(\frac{\partial P}{\partial h} \right) A dh$$

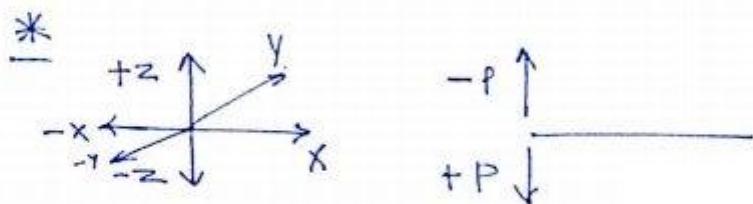
$$\boxed{\frac{\partial P}{\partial h} = \rho g}$$

incompressible fluid

$$\rho \neq f(P)$$

$$dp = \rho g dh$$

$$dp \propto dh$$



$$dp \propto \omega dz$$

↓ ↓
+ve -(-ve)

$$dp = -\omega dz$$

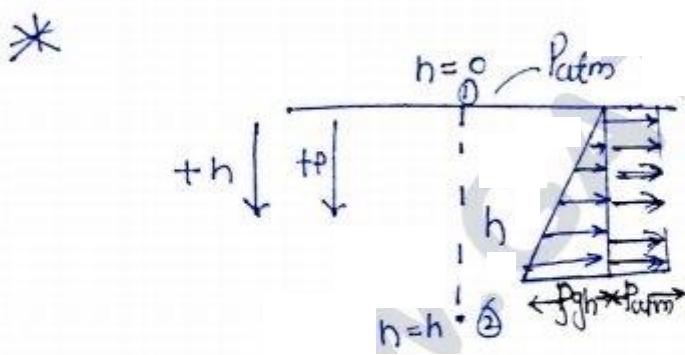
$$\boxed{\frac{dp}{dz} = -\omega = -\rho g}$$

*

$$PA - \left(P + \left(\frac{\partial P}{\partial x} \right) dx \right) A = 0$$

$$\frac{\partial P}{\partial x} = 0$$

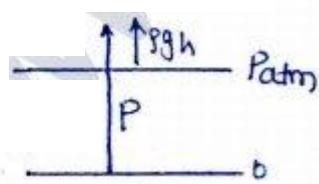
Pressure remains constant at Detum,



$$\frac{dp}{dh} = \rho g$$

incompressible flow

$$\frac{1}{\rho g} \int_{Patm}^P dp = \int_0^h dh$$

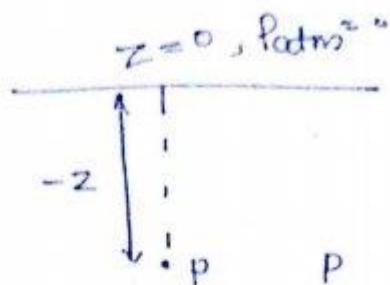
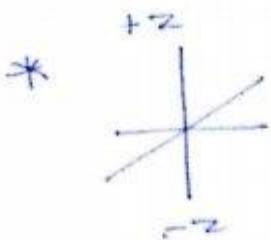


$$P = Patm + \rho gh$$

Absolute

$$P = \rho gh$$

Gauge.
(Patm = 0)



$$\frac{dp}{dz} = -\omega \Rightarrow \int_0^p dp = -\rho g \int_0^z dz$$

~~P = \rho gh~~ ~~P = \rho c g h~~ Gauge

$$P = \rho g z \quad \text{Gauge pressure}$$

Gas (in Compressible fluid)

$$dp = \rho g dh$$

ideal Gas

$$P = \rho R T \quad \leftarrow \text{const.}$$

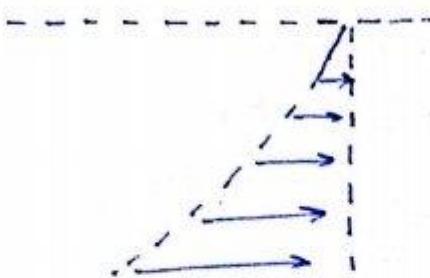
$$dp = \frac{P}{R T} \rho g dh$$

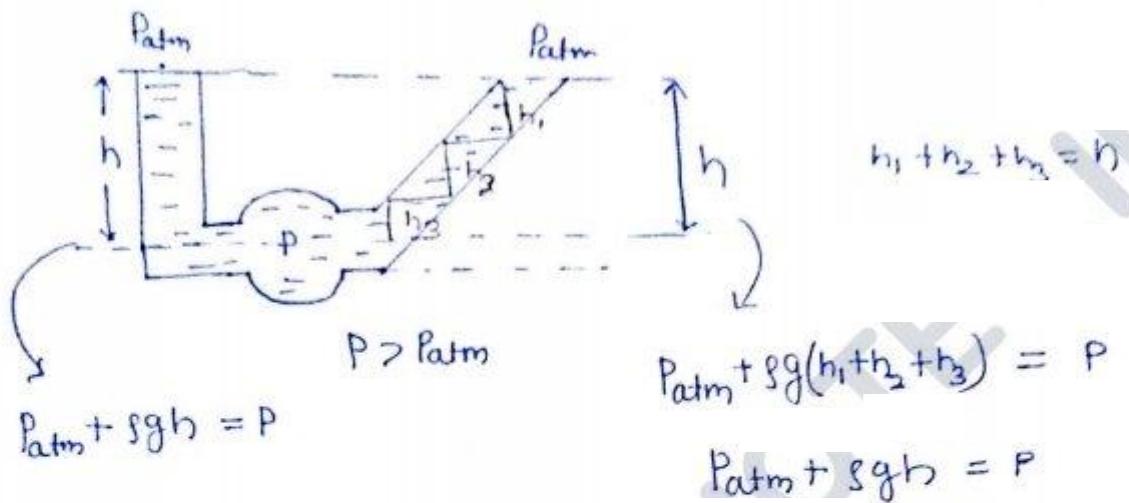
$$\int_{P_{atm}}^P \frac{dp}{P} = \int_0^h \frac{\rho g}{R T} dh$$

$$\rho = \frac{P}{R T}$$

$$\ln \left(\frac{P}{P_{atm}} \right) = + \frac{gh}{RT}$$

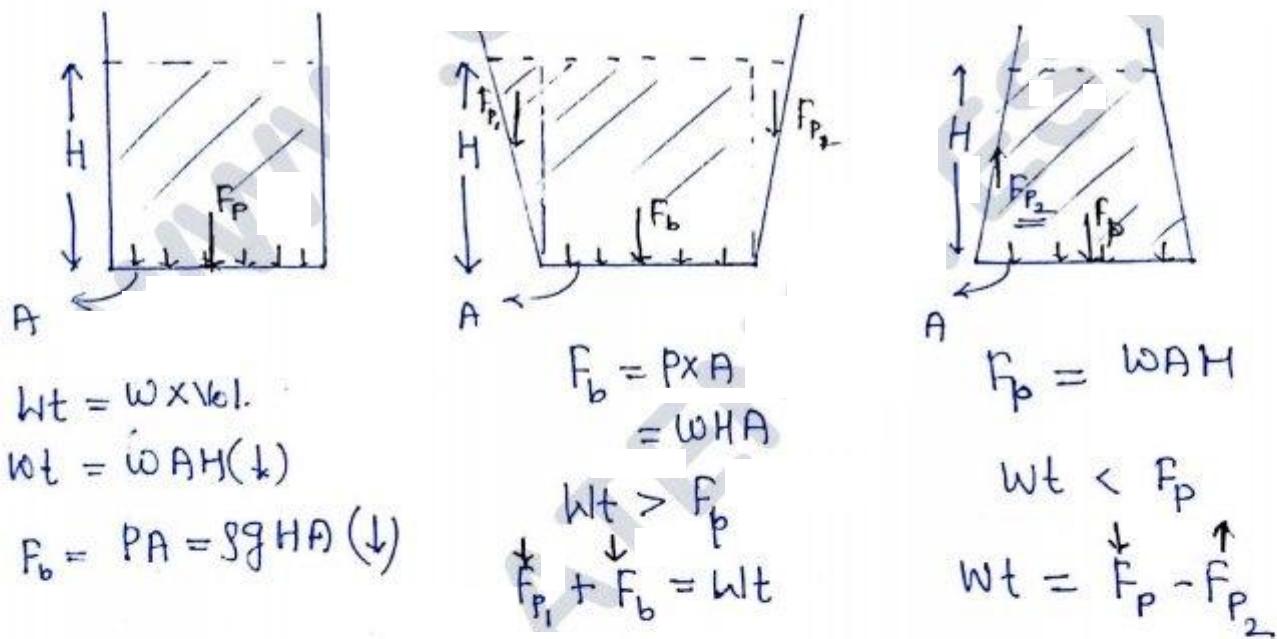
$$P = P_{atm} e^{\frac{gh}{RT}}$$





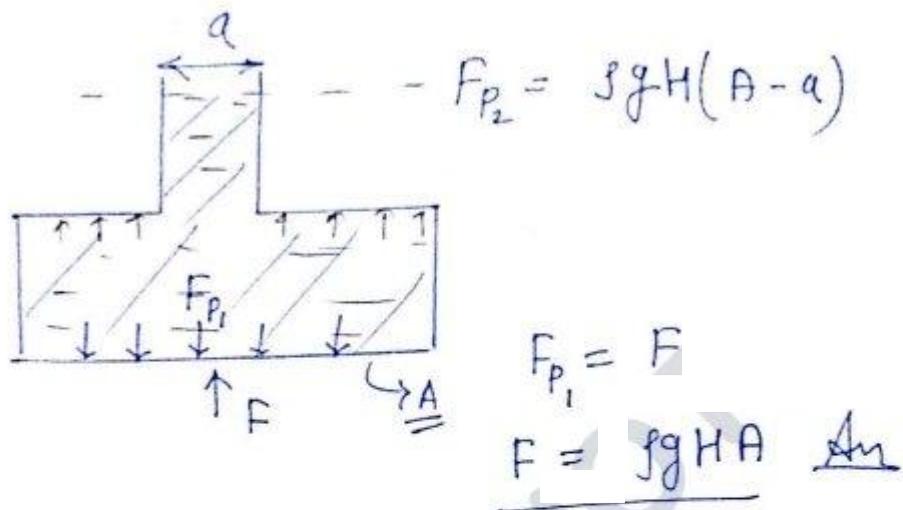
Pressure remain same on datum.

Hydrostatic Paradox:-



- * Hydrostatic force depends on the depth of container and surface area only for the given fluid.

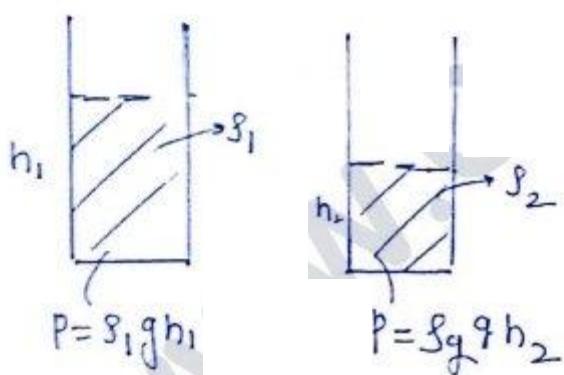
Q.3



$$F_{p1} = F$$

$$\underline{F = fgHA \quad \text{Ans}}$$

Conversion of one liquid column to another:-



$$P = P$$

$$s_1 g h_1 = s_2 g h_2$$

$$s_1 h_1 = s_2 h_2$$

$$h_2 = \frac{s_1}{s_2} h_1$$

$$s_1 g h_1 = s_2 g h_2$$

$$\omega_1 h_1 = \omega_2 h_2$$

$$h_2 = \frac{\omega_1 h_1}{\omega_2}$$

$$\frac{s_1 h_1}{s_w} = \frac{s_2 h_2}{s_w}$$

$$s_1 h_1 = s_2 h_2$$

$$h_2 = \frac{s_1 h_1}{s_2}$$

If $s_2 = 1$ (water.)

$$h_2 = s_1 h_1$$

Atmospheric Pressure

The pressure exerted by the environmental mass on earth surface is known as atmospheric pressure.

$$P_{atm} = 101.325 \text{ kPa}$$

$$P_{atm} = 101.325 \text{ kPa}$$

water
earth

h_w { water
water

$$\omega_w h_w = P_{atm}$$

$$9810 \times h_w = 101.325 \times 10^3 \text{ Pa}$$

$$h_w = 10.3 \text{ m of water}$$

h_{Hg} { Hg

$$\omega_{Hg} h_{Hg} = 101.325 \times 10^3$$

$$h_{Hg} = \frac{101.325 \times 10^3}{13.6 \times 9810}$$

$$h_{Hg} = 760 \text{ mm of Hg}$$

$$h_{Hg} = 76 \text{ cm of Hg.}$$

Units of Pressure:-

$$P = \frac{F}{A} \quad \text{N/m}^2 \quad (\text{S.I.})$$

$$P_{atm} = 101.325 \times 10^3 P_1$$

$$\text{N/m}^2 = \text{Pa}$$

$$10^3 \text{ N/m}^2 = \text{KPa}, \quad 10^6 \text{ N/m}^2 = \text{MPa}$$

$$P_{atm} = 101.325 \times 10^3 Pa = 101325 \times 10^5 Pa$$

$$\approx 1 \times 10^5 Pa \approx 1 \times 10^5 Pa$$

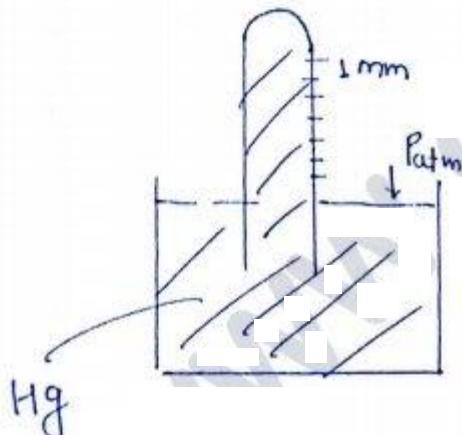
$$10^5 Pa = \frac{kg f}{cm^2}$$

$$10^5 Pa = 1 bar$$

$$1 atm = 1 bar$$

$$1 bar = \frac{kg f}{cm^2}$$

Barometer. (By Torricelli)



$$1 Torr = 1 mm of Hg$$

$$1 Torr = 10^{-3} m of Hg$$

$$= 10^{-3} \times 13.6 \times 9810$$

$$1 Torr = 133.33 Pa.$$

F.P.S. (Foot Pounds Sec.)

$$Psi = \frac{Lbf}{(inch)^2}$$

Pounds Force

Pounds Force
per sq. inch

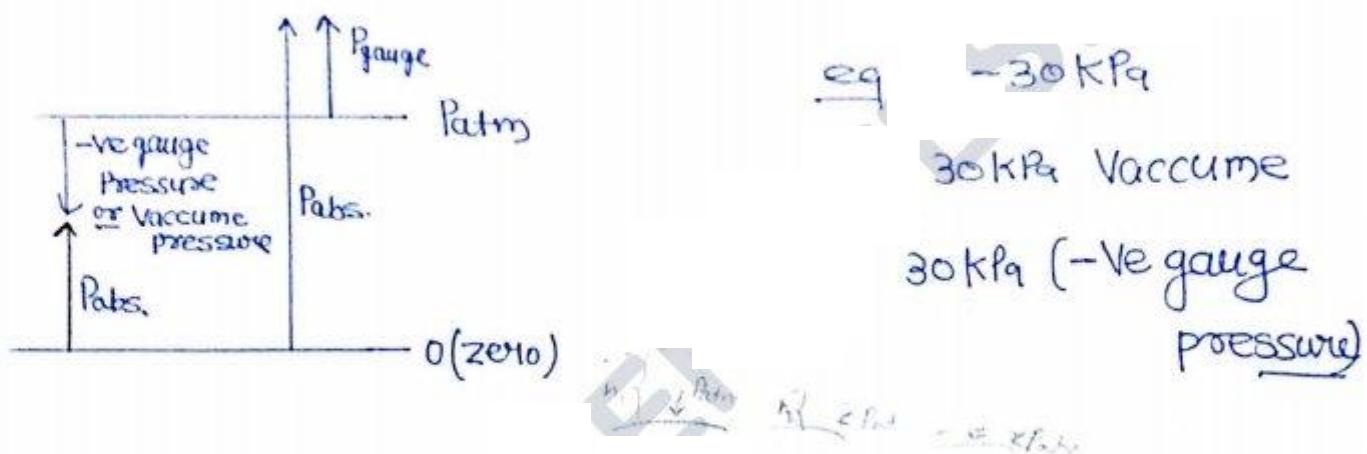
$$Lbf = 0.453 kg f$$

$$Lbf = 0.453 \times 9.81 N$$

$$1 inch = 2.54 cm$$

$$1 atm = 14.7 Psi$$

Absolute Pressure, Gauge Pressure, -ve gauge Pressure
Vacuum pressure.



There may be positive gauge pressure or negative gauge pressure but there can not be negative absolute pressure.

Gauges are calibrated as per their local envt.

Pressure measured from atmospheric pressure is gauge pressure and pressure measured from zero pressure is absolute pressure.

Classification of pressure measurement device.

① on the basis of liquid column deflection (Manometers)

(i) simple manometers

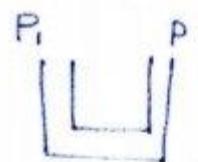


(a) Piezometers

(b) U-tube manometer

(c) single column manometers

(ii) Differential Manometers



(a) U-tube differential manometer

(b) Inverted tube manometer.

② on the basis of elastic properties.
(Mechanical Gauge)

\Downarrow
Bourdon tube
Gauge

Piezometers:- It is the simple manometer if one end open to atmosphere and another end is connected where pressure is to measure.

Disadvantages: (i) Can not measure Gases pressure
(ii) Can not measure Vacuum pressure

(iii) Gives large deflection at high pressure for lighter liquids so moderate pressure should be measured.

Connected with pipeline

$$P - \gamma gh = P_{atm}$$

$$P = P_{atm} + \gamma gh$$

↑ Abs.

$$P_{atm} = 0$$

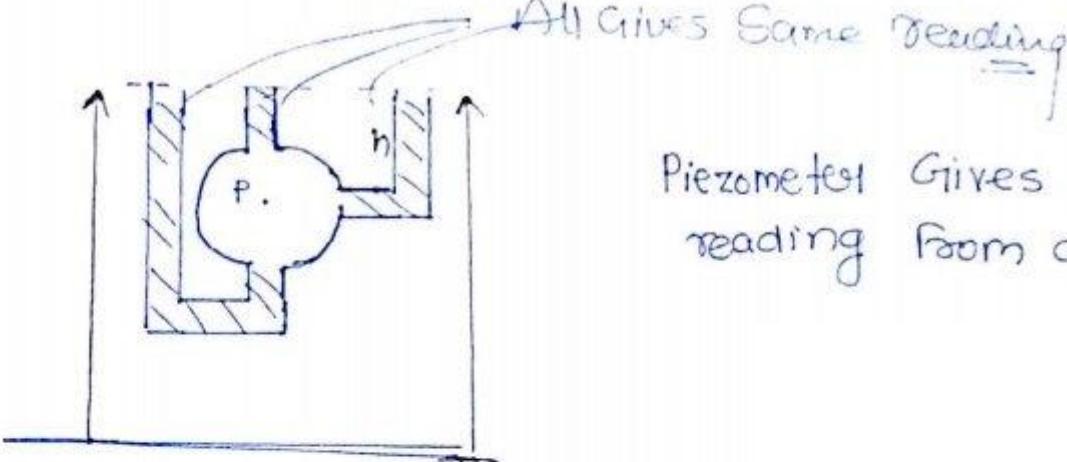
$$P = \gamma gh \text{ Gauge}$$

$$P = \omega h$$

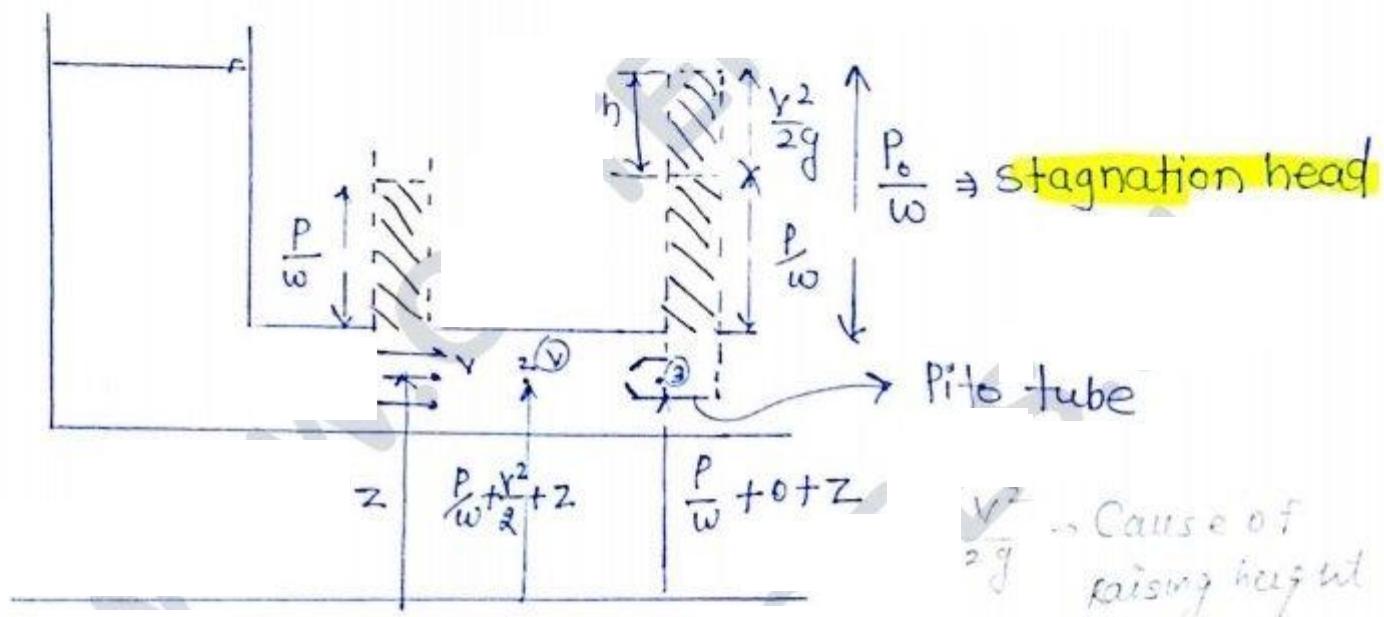
$$h = \frac{P}{\omega}$$

* Piezometric head $= \frac{P}{\omega} + z$

static head datum head



Piezometer Gives same reading from datum



$$h = \frac{V^2}{2g} \Rightarrow V = \sqrt{2gh}$$

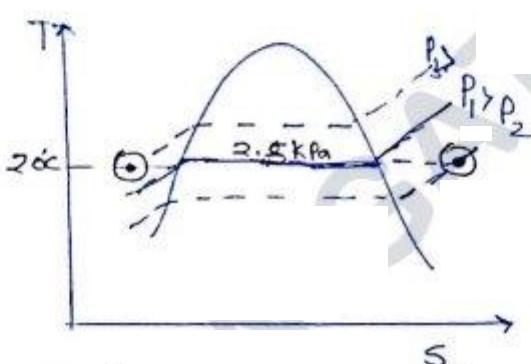
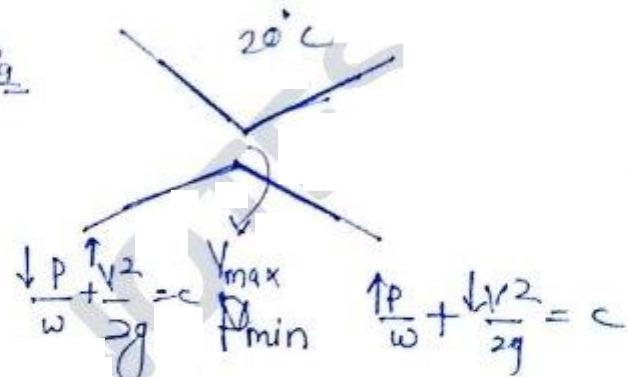
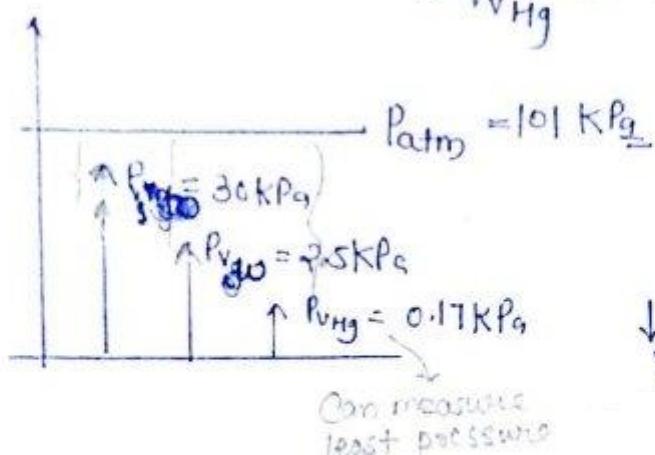
Vapour Pressure: \rightarrow Pressure exerted (partial pressure) by the saturated vapour in the close container on the liquid surface at given temperature is known as saturated vapour pressure.

$$\text{at } 20^\circ\text{C} \quad P_{V,\text{water}} = 2.5 \text{ kPa}$$

$$\text{at } 100^\circ\text{C} \quad P_{V,\text{water}} = 100 \text{ kPa} \approx 1 \text{ atm}$$

at 20°C Gasoline $P_{Vg} = 30 \text{ kPa}$

$$\text{Ans} P_{VHg} = 0.17 \text{ kPa}$$



→ at 20° above P-Vapour
sat liquid

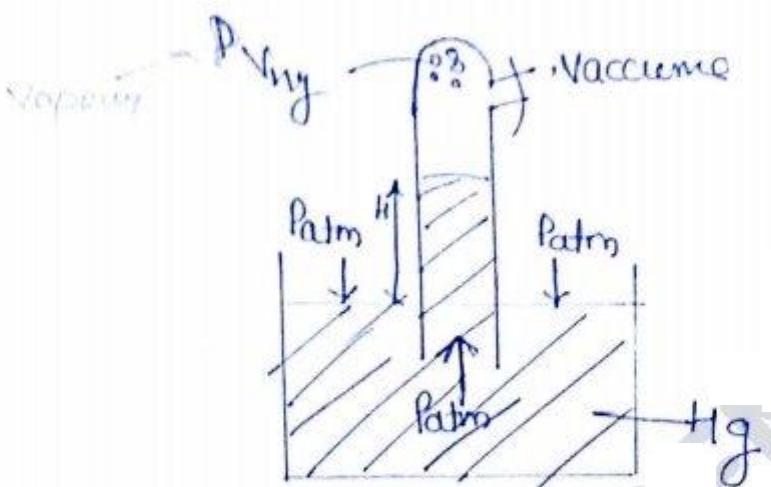
→ at 20° below P-Vapour
saturated steam.

* of const. temp if P below P-vapour
then fluid (h₂o, water) evaporation
bubble forms

Cavitation: In the fluid if the abs pressure of fluid becomes less than corresponding temp. Vapour pressure the bubbles are formed and if these bubbles comes in a region of high pressure the bubbles gets collapse the pitting, cavitation phenomena takes place it erode the surface.

~~eg. at succession side of pump, exit of reaction turbine etc.~~

Barometer:- it is an instrument which measures local atmospheric pressure

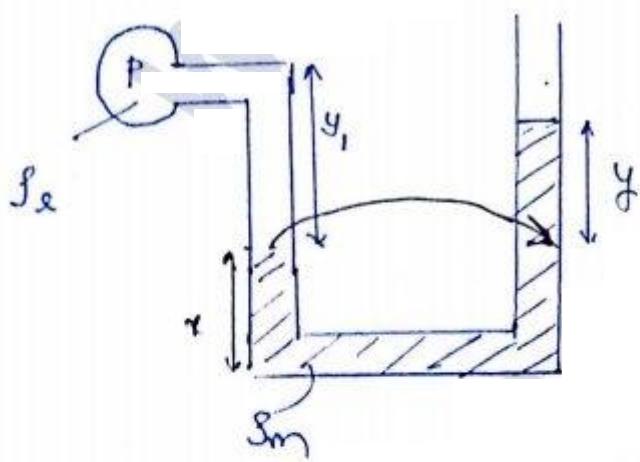


$$P_{v(Hg)} + \rho_{Hg} g H = P_{atm}$$

$$\boxed{P_{atm} = \rho_{Hg} g H}$$

* Mercury (Hg) used in barometers because of high density and low vapour pressure.

U-Tube manometer:-

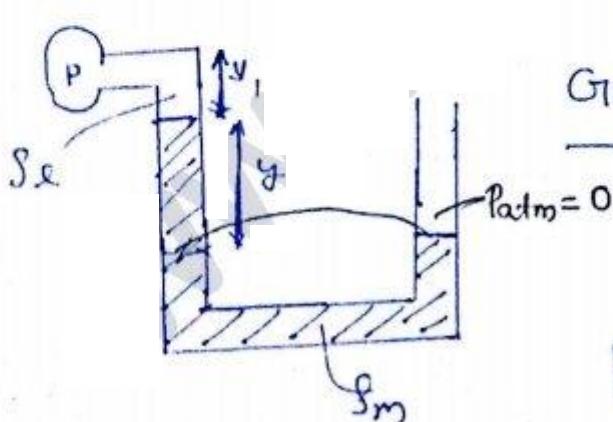


$$P + \rho_e g y_1 + \rho_m g x - \rho_m g x = P_{atm}$$

$$- \rho_m g y = P_{atm}$$

$$P = P_{atm} + \rho_m g y - \rho_e g y_1$$

↳ Absolute



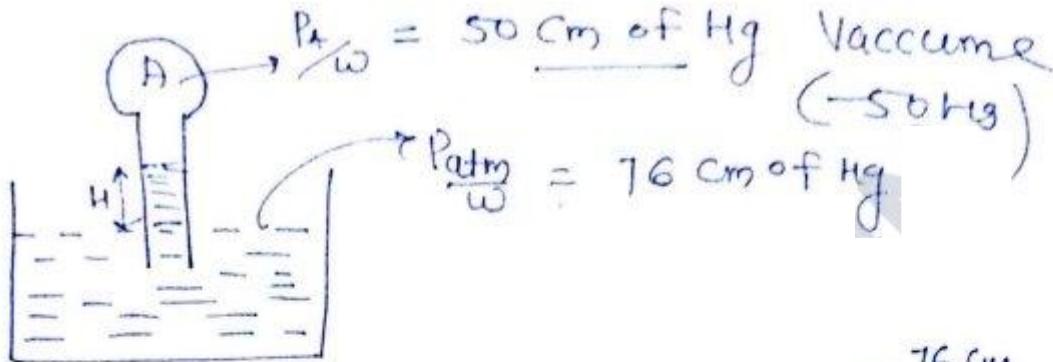
$$P_{atm} = 0$$

$$\text{Gauge} \Rightarrow P = \rho_m g y - \rho_e g y_1$$

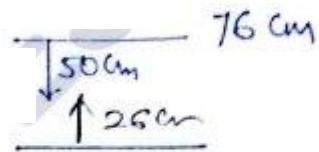
$$0 - \rho_m g y - \rho_e g y_1 = P$$

$$P = - \rho_m g y - \rho_e g y_1 \quad \text{Gauge}$$

Q.3
Q.12



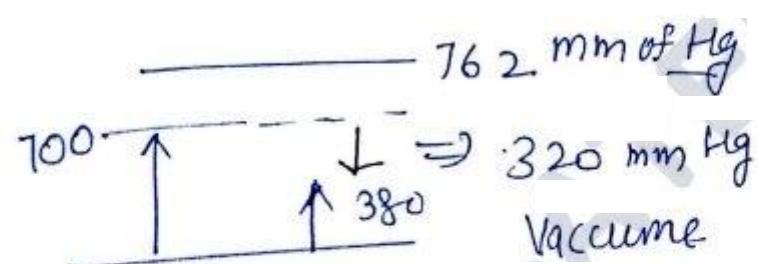
$$\frac{P_A}{w} + H = 76$$



Also. ~~pressure~~ $26 + H = 76$
 $H = 50 \text{ cm of Hg}$ Aw

or Consider $P_{\text{atm}} = 0$
 $0 - H = -50 \Rightarrow \text{Gauge}$
 $H = 50 \text{ cm}$

Q.13 $P_{\text{atm}} = 762 \text{ mm of Hg}$

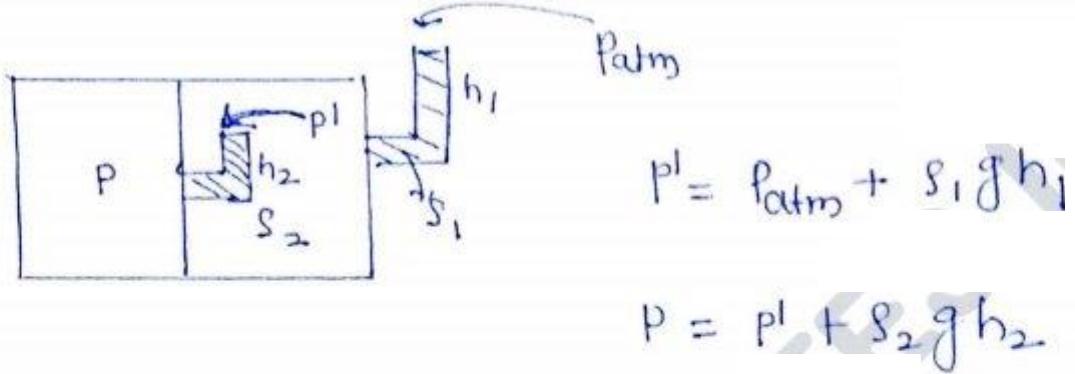


760 mm locat envt.
reading.

$$(P)_{\text{Vac}} = 760 - 380 = 320 \text{ mm Hg} \text{ Vacuum}$$

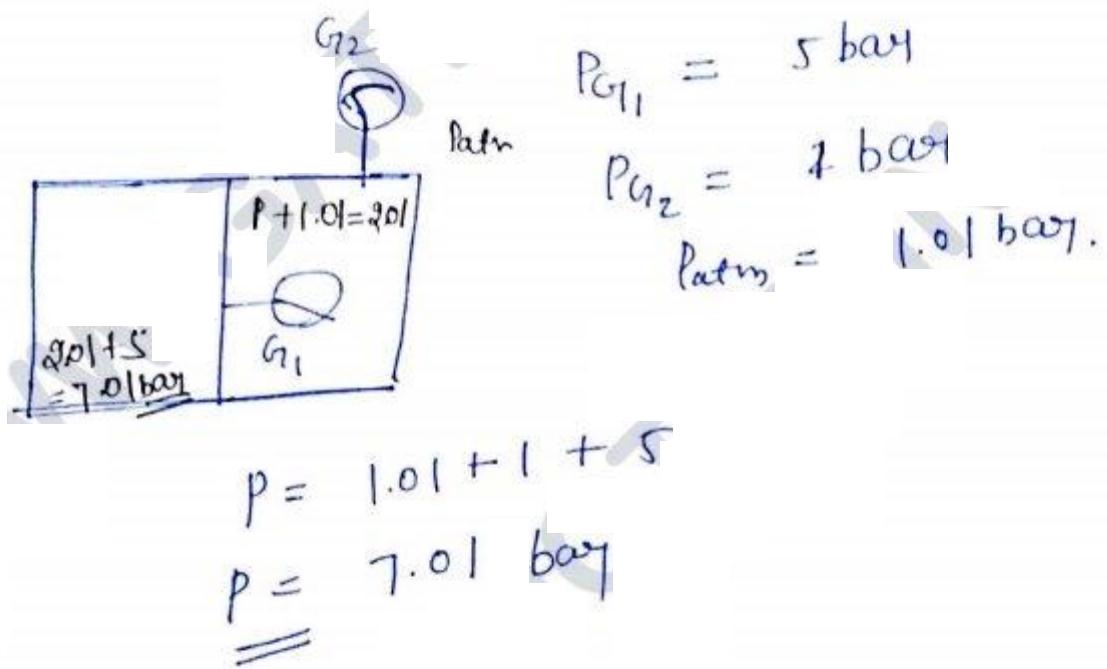
Q.14

*



$$p = p_{atm} + s_1 g h_1 + s_2 g h_2$$

Q. 19



$$p_{G11} = 5 \text{ bar}$$

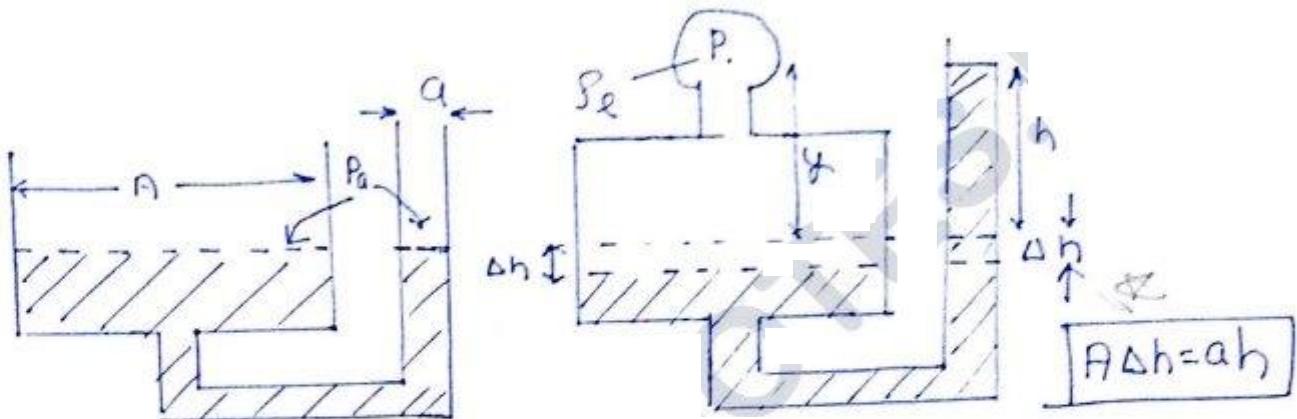
$$p_{G12} = 1 \text{ bar}$$

$$p_{atm} = 1.01 \text{ bar}$$

$$p = 1.01 + 1 + 5$$

$$p = 7.01 \text{ bar}$$

Single Column Manometer:-



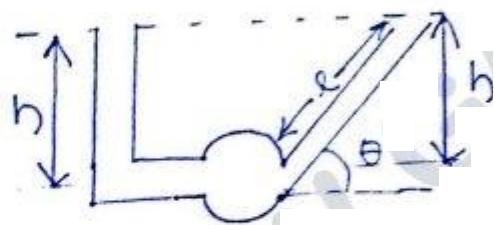
$$P + \rho_l g y + \rho_l g \Delta h = \rho_m g h - \rho_m g h = P_{atm}$$

Q14 $A = 500 \text{ a}$ error = $(h + \Delta h) - h$

$A \Delta h = ah$

$\% \text{ error} = \frac{\Delta h}{h} \times 100 = \frac{a}{A} \times 100 = \frac{100}{500} = 0.2\%$

Sensitivity of manometer :-

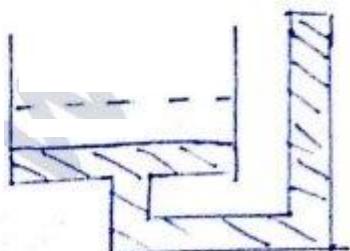


$$l \sin \theta = h$$

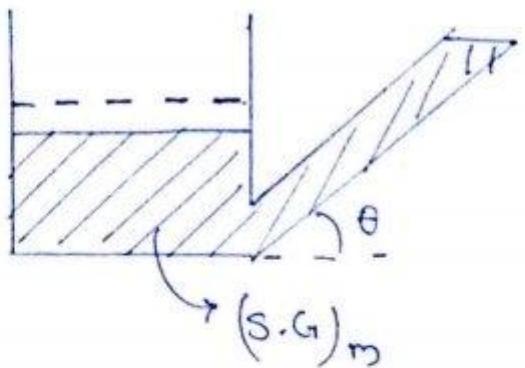
$$\text{sensitivity } (S \propto \frac{l}{h})$$

$$S \propto \frac{1}{\sin \theta}$$

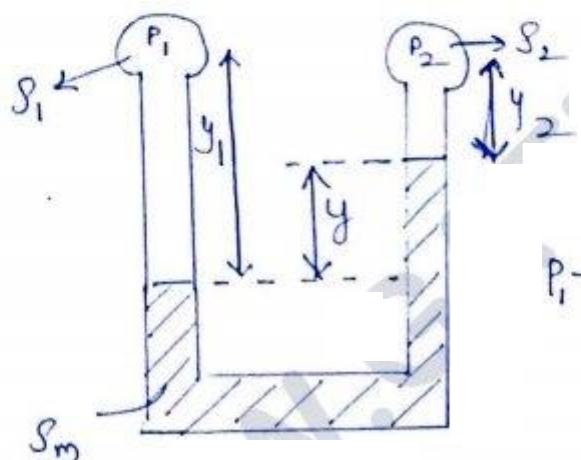
$$\theta \downarrow \rightarrow \sin \theta \downarrow \rightarrow S \uparrow$$



$$S \propto \frac{1}{(\rho_A)} \quad , \quad S \propto \frac{1}{(S.G)_m}$$



For differential manometer

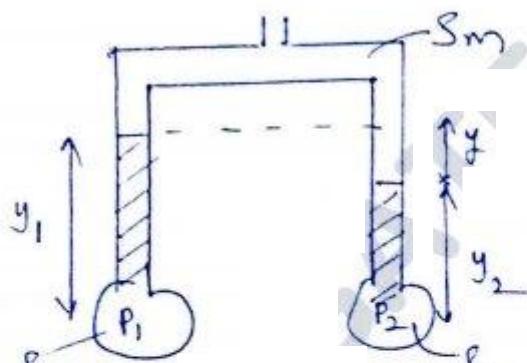


$$P_1 + S_1 g y_1 - S_m g y - S_2 g y_2 = P_2$$

$$P_1 - P_2 = S_m g y - S_1 g y_1 + S_2 g y_2$$

Inverted tube manometry

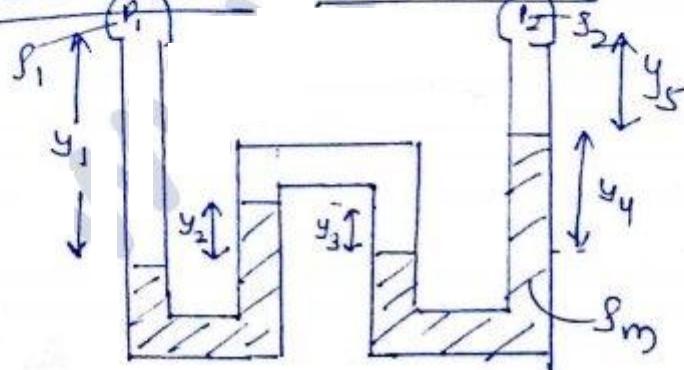
Used for to measure low pressure diff.



$$P_1 - S_1 g y_1 + S_m g y + S_2 g y_2 = P_2$$

$$P_1 - P_2 = -S_m g y + S_1 g y_1 - S_2 g y_2$$

Multi U-tube manometer:- for high pressure diff



$$P_1 - P_2 = -S_1 g y_1 + S_m g y_2$$

$$\begin{aligned} & - S_m g y_3 + S_m g y_4 \\ & + S_2 g y_5 \end{aligned}$$